
Connecting Continuous and Discrete Systems using Control Volume

Edward Smith

Collaborating with:
D. M. Heyes, D. Dini, T. A. Zaki

Mechanical Engineering
Imperial College London

Outline

- **Introduction**

- Continuum Mechanics & Molecular Dynamics
- Selecting function and Irving and Kirkwood (1950)
- Control Volume form

- **The Control Volume Function**

- Discrete Reynolds' transport theorem
- Governing equations
- Applying the method to state coupling

- **Flux Coupling and the Pressure Tensor**

- Derivation of the flux form of the control volume equations
- Pressure tensor
- Coupling fluxes

Introduction

Continuum vs. Discrete

- Assumed Continuous at every point in space

- Mass Conservation

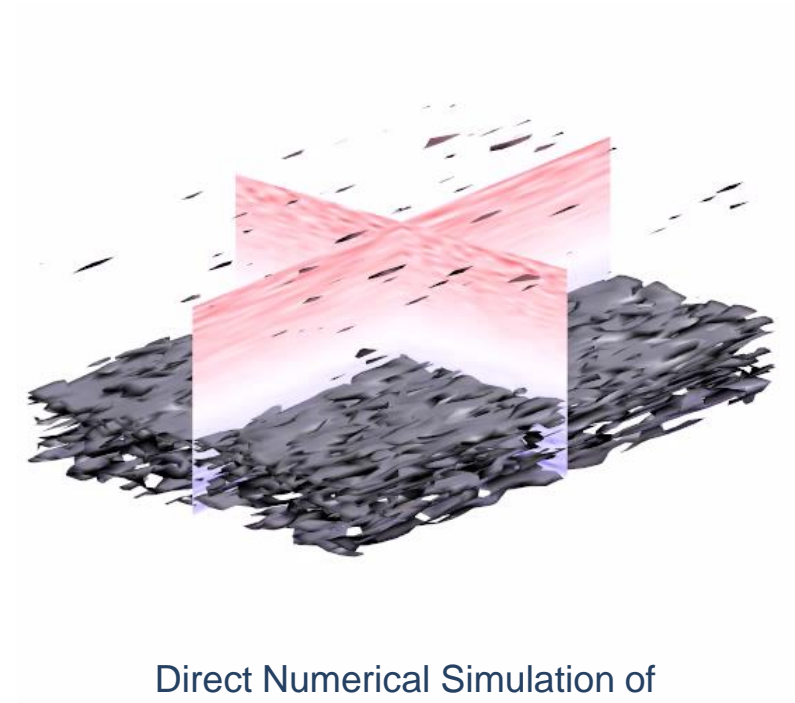
$$\frac{\partial \rho}{\partial t} = -\nabla \cdot \rho \mathbf{u}$$

- Momentum Balance (Newton's Law)

$$\frac{\partial}{\partial t} \rho \mathbf{u} + \nabla \cdot \rho \mathbf{u} \mathbf{u} = \nabla \cdot \mathbf{\Pi}$$

- Energy Conservation

$$\frac{\partial}{\partial t} \rho \mathcal{E} dV = -\nabla \cdot [\rho \mathcal{E} \mathbf{u} + \mathbf{\Pi} \cdot \mathbf{u} + \mathbf{q}]$$



Direct Numerical Simulation of
Turbulent Couette Flow

Continuum vs. Discrete

- Discrete Molecules in continuous space

- Governed by Newton's Law for an N-body system
- Point particles with pairwise interactions

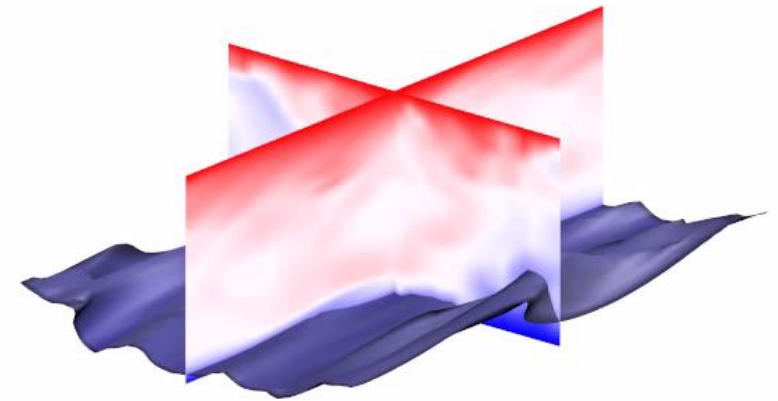
$$m_i \ddot{\mathbf{r}}_i = \mathbf{F}_i = \sum_{i \neq j}^N \mathbf{f}_{ij}$$

- Average required to obtain the continuous field

- How do we get an Eulerian description?

$$\rho = \frac{1}{V} \sum_{i=1}^{M_I} m_i$$

where M_I is the number of molecules in volume V

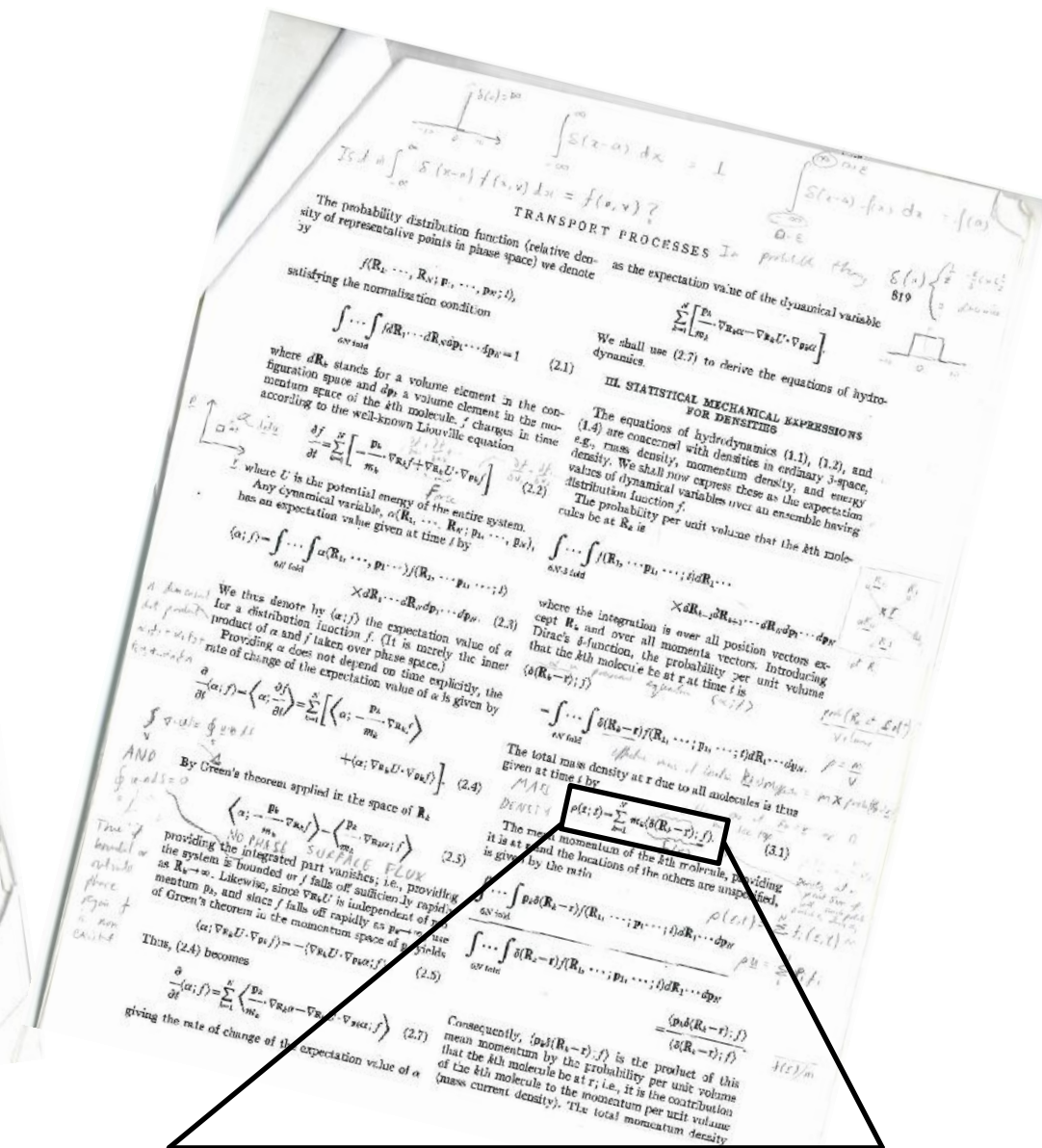
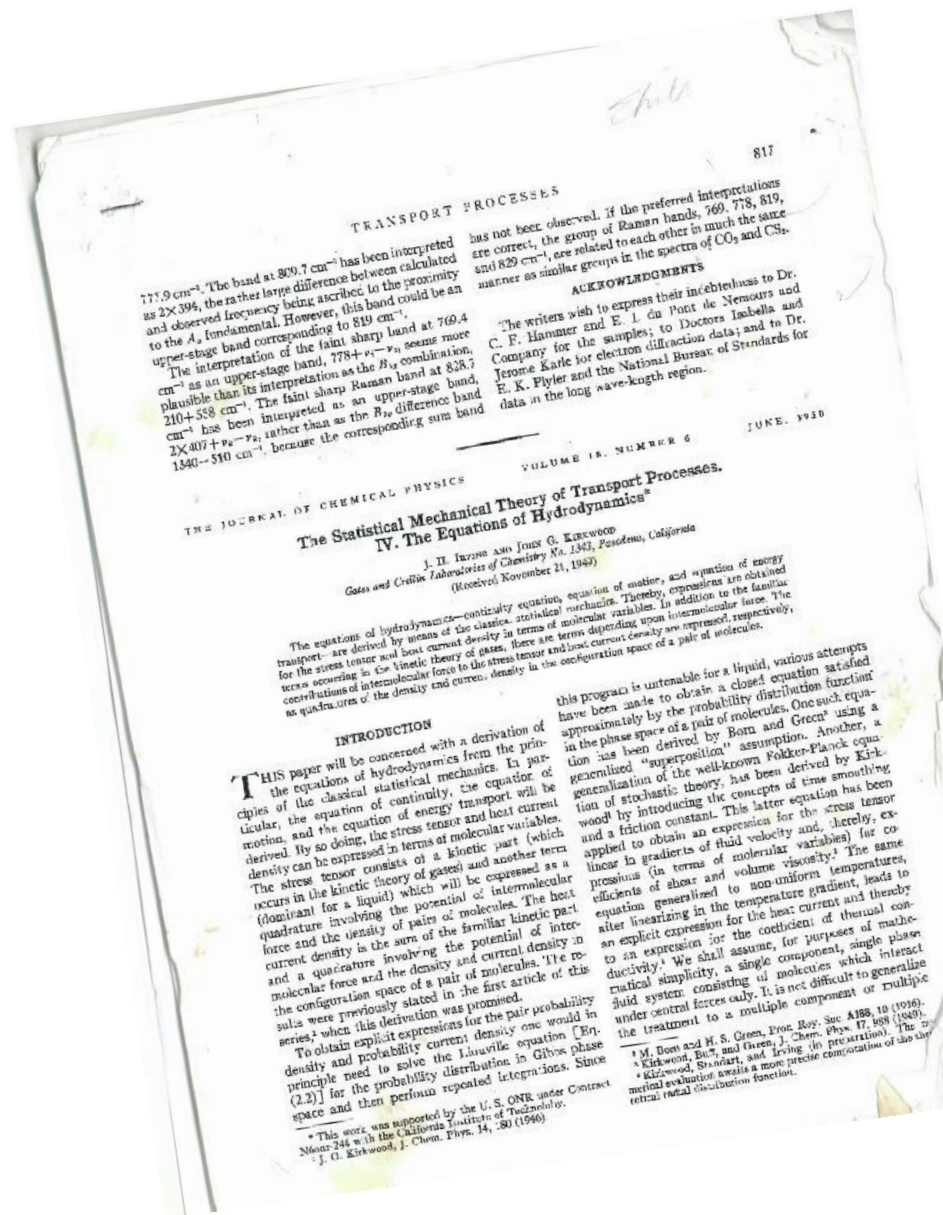


Molecular Dynamics Simulation of Couette Flow

$$\rho = \frac{1}{V} \sum_{i \in S} m_i$$

where $S = \{i | \mathbf{r}^- < \mathbf{r}_i < \mathbf{r}^+\}$

Irving and Kirkwood (1950)



Selecting Functions

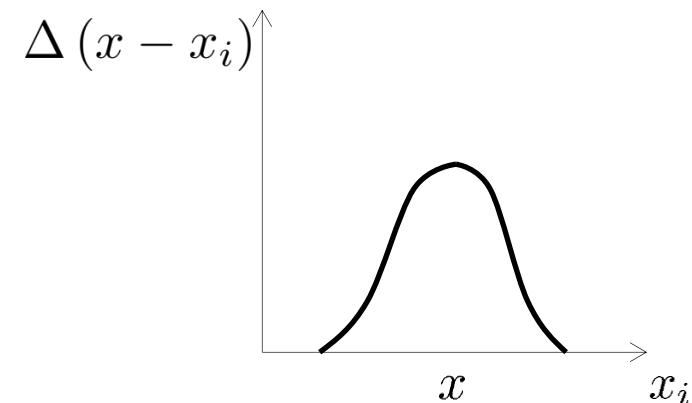
- **The Dirac delta selects molecules at a point**

- Infinitely high, infinitely thin peak
- Equivalent to the continuum differential formulation at a point

$$\rho(\mathbf{r}, t) = \sum_{i=1}^N m_i \delta(\mathbf{r} - \mathbf{r}_i)$$

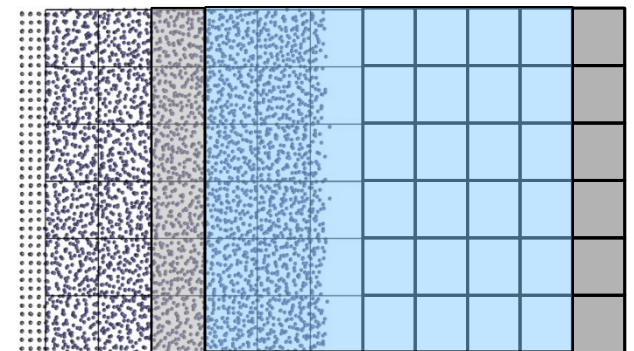
- **Cannot be applied directly in a molecular simulation as \mathbf{r}_i is never exactly equal to \mathbf{r}**

- Relaxed weighting function used*



- **Co-existence of two descriptions in same simulation**

- Overlap used in most coupling schemes
- Length scales comparable at interface



The Control Volume (CV)

- A finite volume with fluxes and forces acting over its surfaces

- Mass Conservation

$$\frac{\partial \rho}{\partial t} = -\nabla \cdot \rho \mathbf{u}$$

$$\frac{\partial}{\partial t} \int_V \rho dV = - \oint_S \rho \mathbf{u} \cdot d\mathbf{S}$$

- Momentum Balance (Newton's Law)

$$\frac{\partial}{\partial t} \rho \mathbf{u} + \nabla \cdot \rho \mathbf{u} \mathbf{u} = \nabla \cdot \mathbf{\Pi}$$

$$\frac{\partial}{\partial t} \int_V \rho \mathbf{u} dV = - \oint_S \rho \mathbf{u} \mathbf{u} \cdot d\mathbf{S} + \mathbf{F}_{\text{surface}}$$

- Energy Conservation

$$\frac{\partial}{\partial t} \rho \mathcal{E} dV = -\nabla \cdot [\rho \mathcal{E} \mathbf{u} + \mathbf{\Pi} \cdot \mathbf{u} + \mathbf{q}]$$

$$\frac{\partial}{\partial t} \int_V \rho \mathcal{E} dV = - \oint_S \rho \mathcal{E} \mathbf{u} \cdot d\mathbf{S} - \oint_S \mathbf{\Pi} \cdot \mathbf{u} \cdot d\mathbf{S} + \mathbf{q} \cdot d\mathbf{S}$$

The Control Volume (CV)

- Writing the molecular system in terms of control volumes

- Mass

$$\rho = \sum_{i=1}^N m_i \delta(\mathbf{r} - \mathbf{r}_i)$$

$$\int_V \rho dV = \sum_{i=1}^N m_i \int_V \delta(\mathbf{r} - \mathbf{r}_i) dV$$

- Momentum

$$\rho \mathbf{u} = \sum_{i=1}^N \mathbf{p}_i \delta(\mathbf{r} - \mathbf{r}_i)$$

$$\int_V \rho \mathbf{u} dV = \sum_{i=1}^N \mathbf{p}_i \int_V \delta(\mathbf{r} - \mathbf{r}_i) dV$$

- Energy

$$\rho \mathcal{E} = \sum_{i=1}^N e_i \delta(\mathbf{r} - \mathbf{r}_i)$$

$$\int_V \rho \mathcal{E} dV = \sum_{i=1}^N e_i \int_V \delta(\mathbf{r} - \mathbf{r}_i) dV$$

The Control Volume Formulation

Control Volume Function

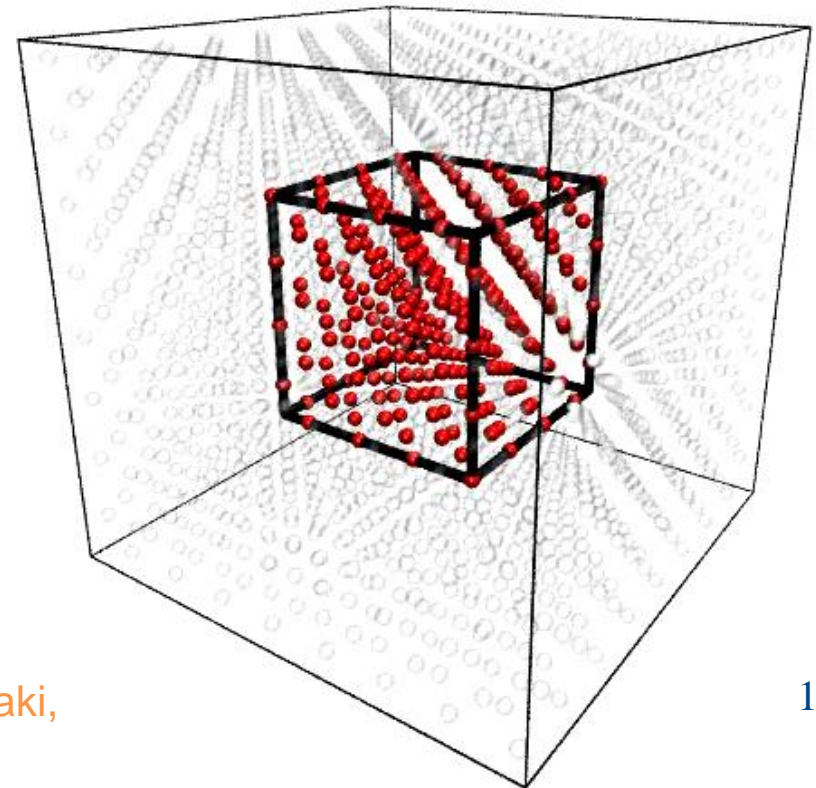
- The Control volume function is the integral of the Dirac delta function in 3 dimensions

$$\vartheta_i \equiv \int_{x^-}^{x^+} \int_{y^-}^{y^+} \int_{z^-}^{z^+} \delta(x_i - x) \delta(y_i - y) \delta(z_i - z) dx dy dz$$

$$= [H(x^+ - x_i) - H(x^- - x_i)]$$

$$\times [H(y^+ - y_i) - H(y^- - y_i)]$$

$$\times [H(z^+ - z_i) - H(z^- - z_i)]$$



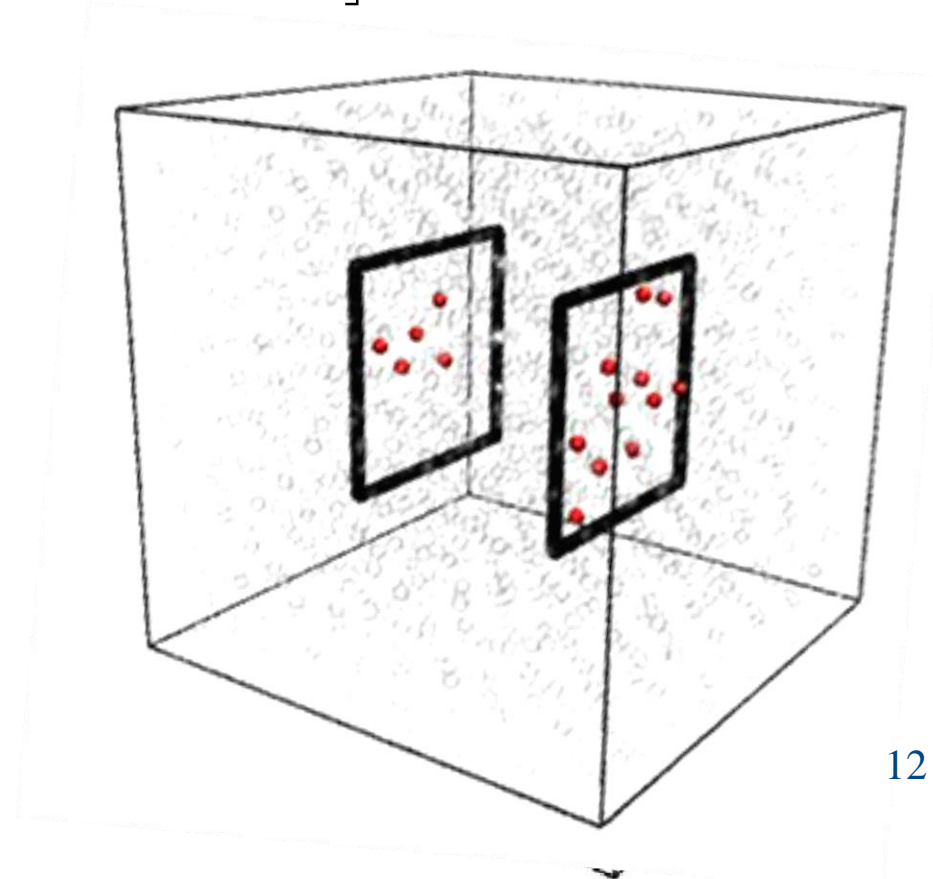
Derivatives yields the Surface Fluxes

- Taking the Derivative of the CV function

$$\begin{aligned} dS_{ix} &\equiv -\frac{\partial \vartheta_i}{\partial x_i} = [\delta(x^+ - x_i) - \delta(x^- - x_i)] \\ &\quad \times [H(y^+ - y_i) - H(y^- - y_i)] \\ &\quad \times [H(z^+ - z_i) - H(z^- - z_i)] \end{aligned}$$

- Surface fluxes over the top and bottom surface

$$dS_{ix} = dS_{ix}^+ - dS_{ix}^-$$



Applying the Control Volume Function

- Molecular mass in a control volume

$$\frac{d}{dt} \sum_{i=1}^N m_i \vartheta_i = - \sum_{i=1}^N m_i \mathbf{v}_i \cdot d\mathbf{S}_i \qquad \frac{\partial}{\partial t} \int_V \rho dV = - \oint_S \rho \mathbf{u} \cdot d\mathbf{S}$$

- Mathematical manipulation yields surface fluxes

$$\begin{aligned} \frac{d}{dt} \sum_{i=1}^N m_i \vartheta_i &= \sum_{i=1}^N \left(\cancel{\vartheta_i \frac{d}{dt} m_i} + m_i \frac{d}{dt} \vartheta_i \right) \\ &= \sum_{i=1}^N m_i \frac{d\mathbf{r}_i}{dt} \cdot \frac{d}{d\mathbf{r}_i} \vartheta_i \\ &= - \sum_{i=1}^N m_i \mathbf{v}_i \cdot d\mathbf{S}_i \end{aligned}$$

Reynolds' Transport Theorem

- Mass, momentum and energy equations

- Mass Conservation

$$\frac{d}{dt} \sum_{i=1}^N m_i \vartheta_i = - \sum_{i=1}^N m_i \mathbf{v}_i \cdot d\mathbf{S}_i$$

$$\frac{\partial}{\partial t} \int_V \rho dV = - \oint_S \rho \mathbf{u} \cdot d\mathbf{S}$$

- Momentum Balance

$$\begin{aligned} \frac{d}{dt} \sum_{i=1}^N m_i \mathbf{v}_i \vartheta_i = & - \sum_{i=1}^N m_i \mathbf{v}_i \mathbf{v}_i \cdot d\mathbf{S}_i \\ & + \sum_{i=1}^N \sum_{j \neq i}^N \mathbf{f}_{ij} \vartheta_{ij} \end{aligned}$$

$$\begin{aligned} \frac{\partial}{\partial t} \int_V \rho \mathbf{u} dV = & - \oint_S \rho \mathbf{u} \mathbf{u} \cdot d\mathbf{S} \\ & + \mathbf{F}_{\text{surface}} \end{aligned}$$

- Energy Conservation

$$\begin{aligned} \frac{d}{dt} \sum_{i=1}^N e_i \vartheta_i = & - \sum_{i=1}^N e_i \mathbf{v}_i \cdot d\mathbf{S}_i \\ & + \frac{1}{2} \sum_{i=1}^N \sum_{i \neq j}^N \frac{\mathbf{p}_i}{m_i} \cdot \mathbf{f}_{ij} \vartheta_{ij} \end{aligned}$$

$$\begin{aligned} \frac{\partial}{\partial t} \int_V \rho \mathcal{E} dV = & - \oint_S \rho \mathcal{E} \mathbf{u} \cdot d\mathbf{S} \\ & - \oint_S \mathbf{\Pi} \cdot \mathbf{u} \cdot d\mathbf{S} + \mathbf{q} \cdot d\mathbf{S} \end{aligned}$$

Reynolds' Transport Theorem

- Mass, momentum and energy equations

- Mass Conservation

$$\frac{d}{dt} \sum_{i=1}^N m_i \vartheta_i = - \sum_{i=1}^N m_i \mathbf{v}_i \cdot d\mathbf{S}_i$$

$$\frac{\partial}{\partial t} \int_V \rho dV = - \oint_S \rho \mathbf{u} \cdot d\mathbf{S}$$

- Momentum Balance

$$\begin{aligned} \frac{d}{dt} \sum_{i=1}^N m_i \mathbf{v}_i \vartheta_i = & - \sum_{i=1}^N m_i \mathbf{v}_i \mathbf{v}_i \cdot d\mathbf{S}_i \\ & + \sum_{i=1}^N \sum_{j \neq i}^N \mathbf{f}_{ij} \vartheta_{ij} \end{aligned}$$

$$\frac{\partial}{\partial t} \int_V \rho \mathbf{u} dV = - \oint_S \rho \mathbf{u} \mathbf{u} \cdot d\mathbf{S}$$

+ $\mathbf{F}_{\text{surface}}$

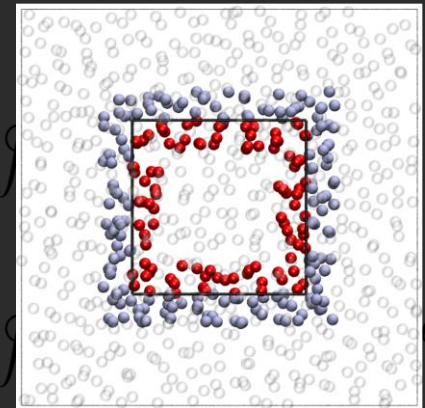
- The difference between two control volume functions for i and j

$$\frac{d}{dt} \sum_{i=1}^N e_i \vartheta_i = - \sum_{i=1}^N e_i \mathbf{v}_i \cdot d\mathbf{S}_i$$

$$\vartheta_{ij} \equiv \vartheta_i - \vartheta_j$$

$$+ \frac{1}{2} \sum_{i=1}^N \sum_{i \neq j}^N \frac{\mathbf{p}_i}{m_i} \cdot \mathbf{f}_{ij} \vartheta_{ij}$$

$$\frac{\partial}{\partial t} \int_V \rho \mathcal{E} dV = - \oint_S \rho \mathcal{E} \mathbf{u} \cdot d\mathbf{S}$$

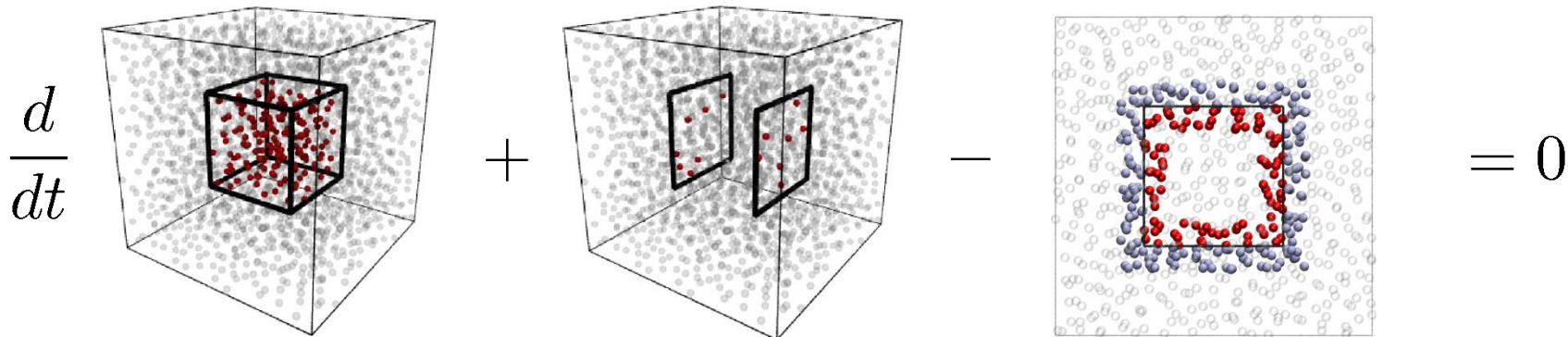


$d\mathbf{S}$

Testing Momentum Balance

Momentum Balance

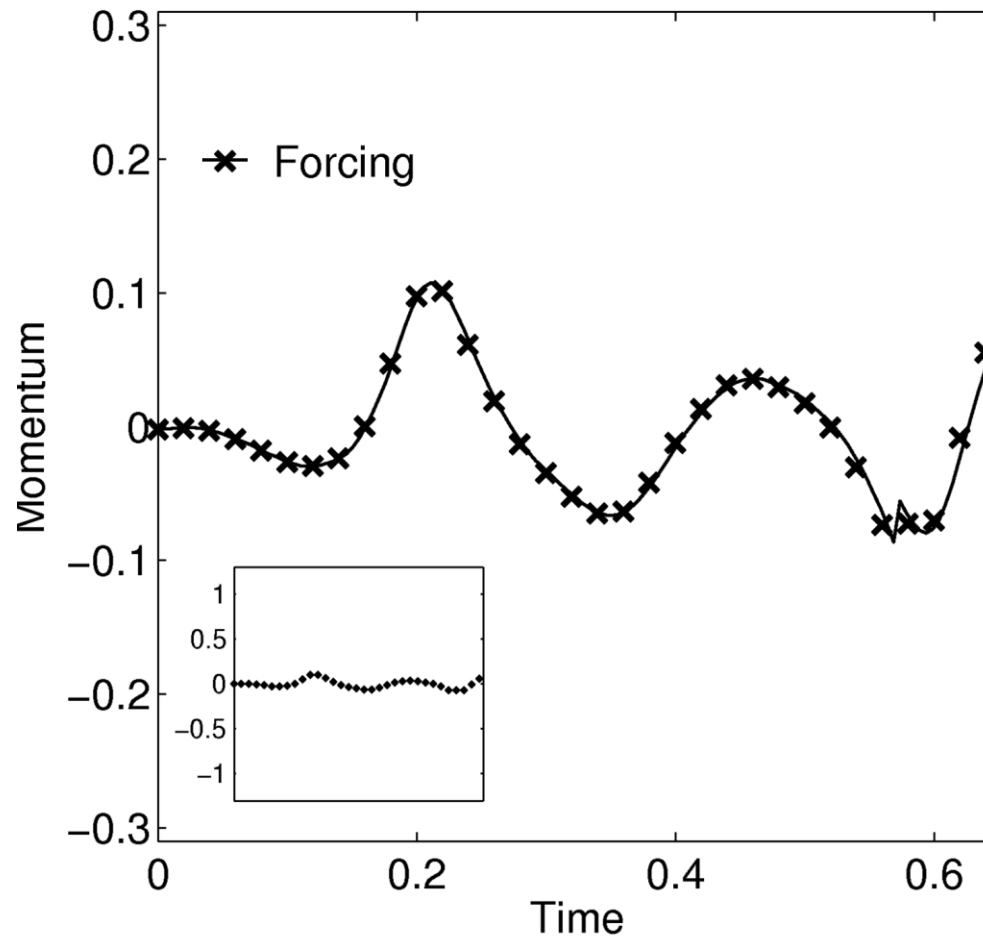
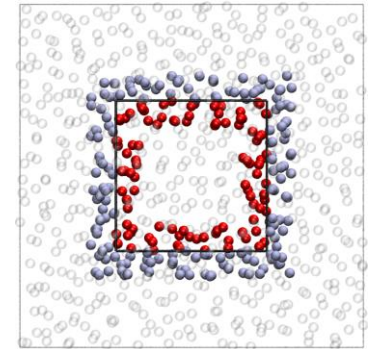
$$\underbrace{\frac{d}{dt} \sum_{i=1}^N m_i \dot{\mathbf{r}}_i \vartheta_i}_{\text{Accumulation}} + \underbrace{\sum_{i=1}^N m \mathbf{v}_i \mathbf{v}_i \cdot d\mathbf{S}_i}_{\text{Advection}} - \underbrace{\sum_{i=1}^N \sum_{j \neq i}^N \mathbf{f}_{ij} \vartheta_{ij}}_{\text{Forcing}} = 0$$



Testing Momentum Balance

- Momentum Balance

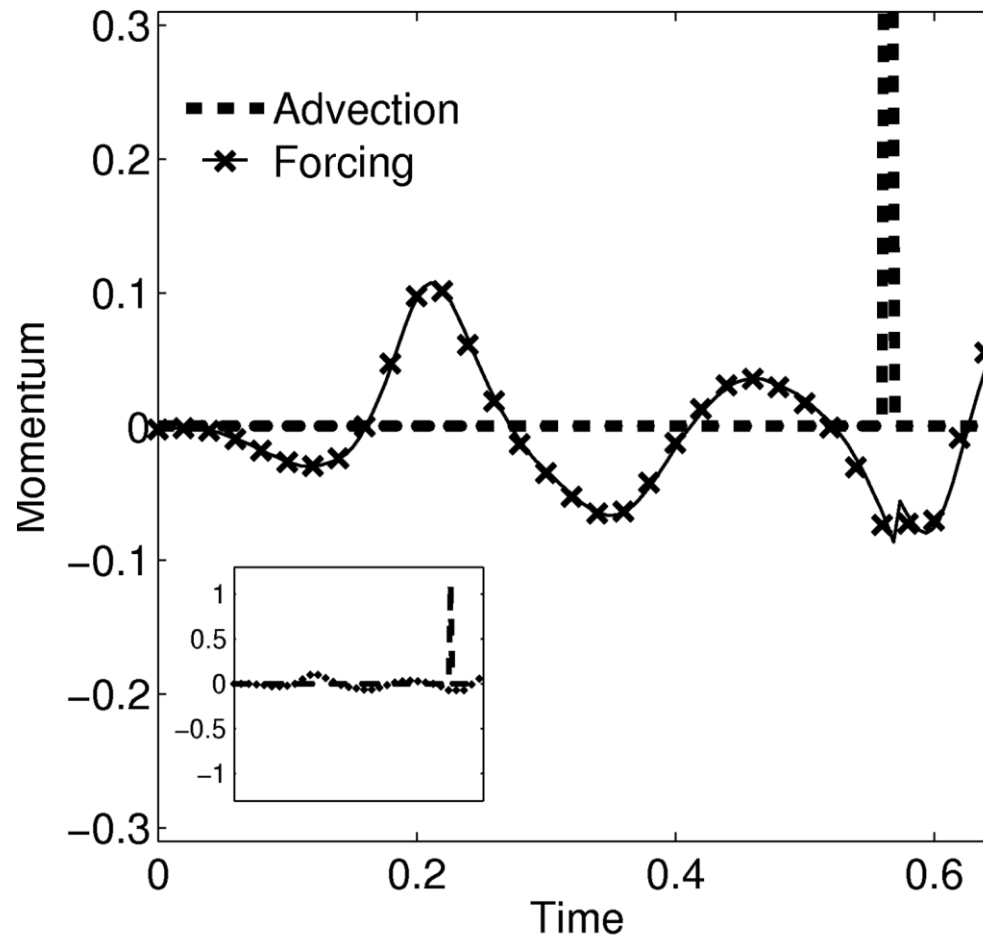
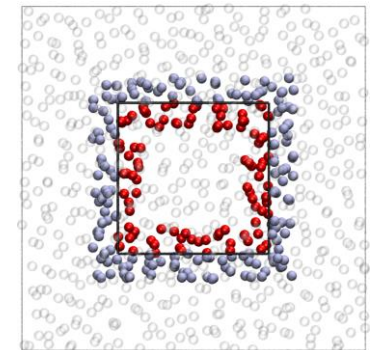
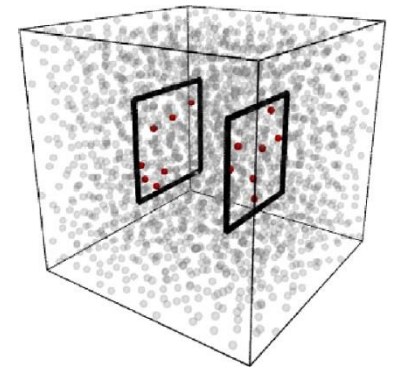
$$\underbrace{\sum_{i=1}^N \sum_{j \neq i}^N \mathbf{f}_{ij} \vartheta_{ij}}_{\text{Forcing}}$$



Testing Momentum Balance

• Momentum Balance

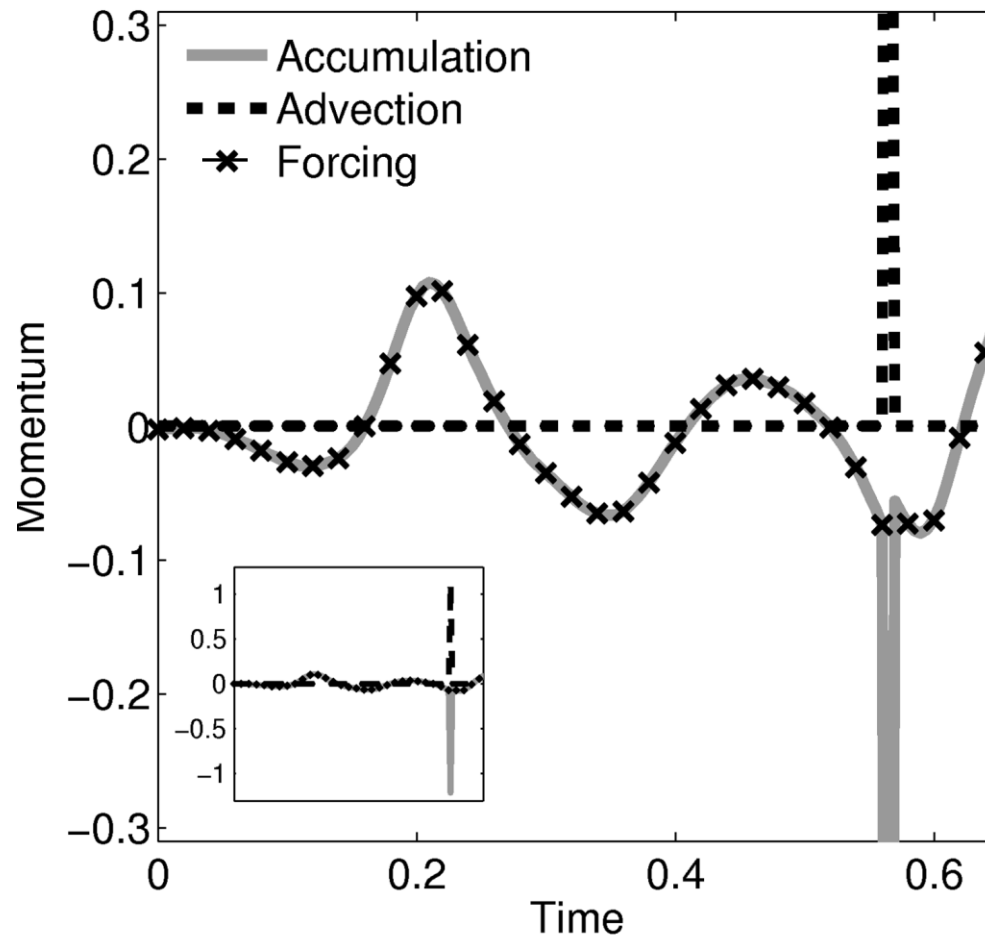
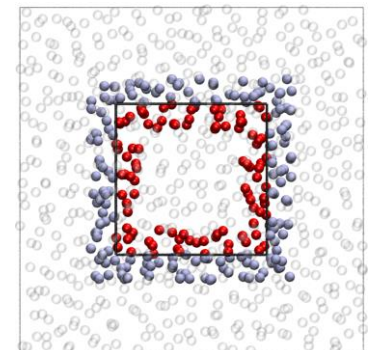
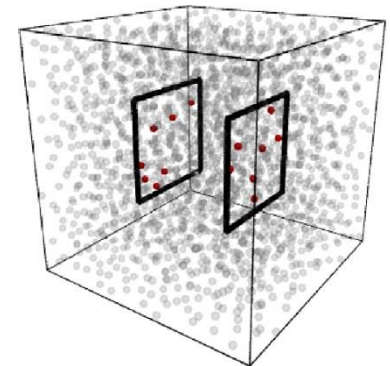
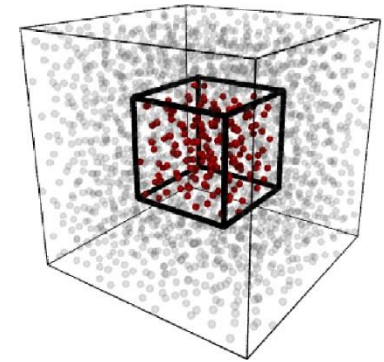
$$-\underbrace{\sum_{i=1}^N m_i \mathbf{v}_i \mathbf{v}_i \cdot d\mathbf{S}_i}_{\text{Advection}} + \underbrace{\sum_{i=1}^N \sum_{j \neq i}^N \mathbf{f}_{ij} \vartheta_{ij}}_{\text{Forcing}}$$



Testing Momentum Balance

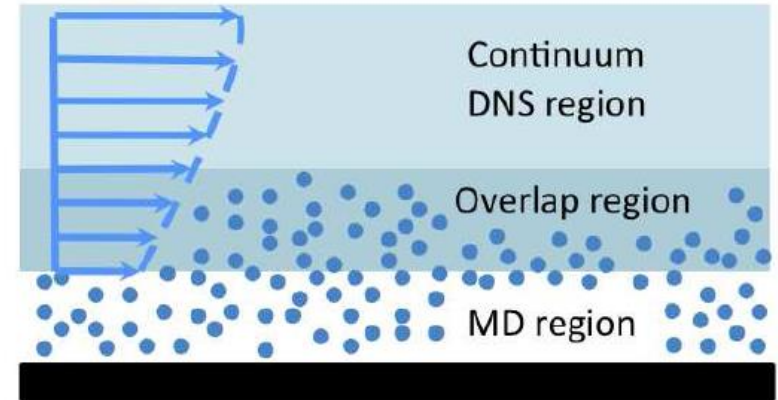
• Momentum Balance

$$\underbrace{\frac{d}{dt} \sum_{i=1}^N m_i \mathbf{v}_i \vartheta_i}_{\text{Accumulation}} = - \underbrace{\sum_{i=1}^N m_i \mathbf{v}_i \mathbf{v}_i \cdot d\mathbf{S}_i}_{\text{Advection}} + \underbrace{\sum_{i=1}^N \sum_{j \neq i}^N \mathbf{f}_{ij} \vartheta_{ij}}_{\text{Forcing}}$$

 $\frac{d}{dt}$


Using the CV for coupling

- **Layout of a typical coupled simulation**
 - Separate MD and CFD regions with Overlap region between
 - MD averages give CFD boundary conditions
 - CFD values used to apply a constraint to MD
- **Control Volume can be used to**
 - Average values from MD
 - Apply a constraint to any arbitrary CV
- **Constraint is non-unique**
 - Hamilton's principle is one of the most fundamental formulations of mechanics
 - Used to apply physically meaningful constraints



$$\int_V \rho \mathbf{u} dV = \sum_{i=1}^N \mathbf{p}_i \vartheta_i$$

$$\delta A = \delta \int_a^b \mathcal{L} dt = 0$$

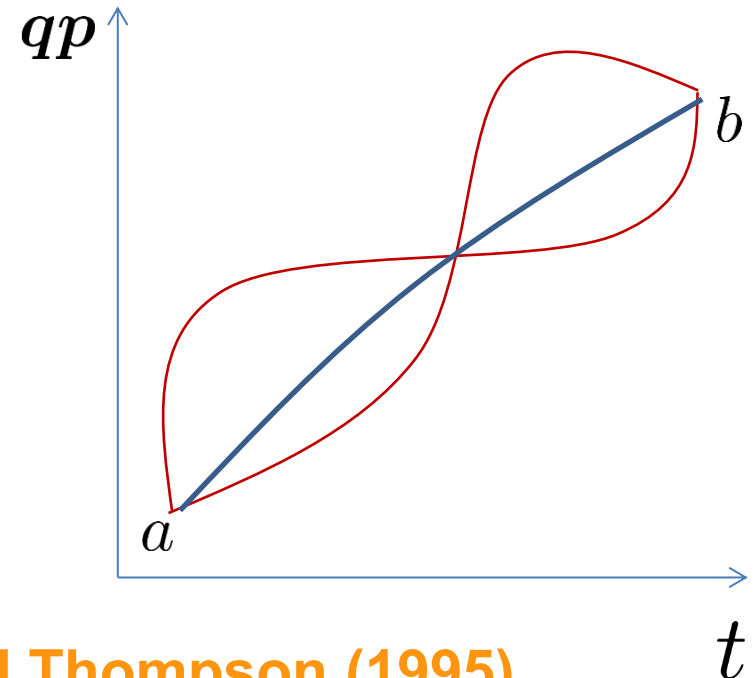
Constraint Algorithms

- Principle of Least Action (subject to constraint)

$$\delta A_c = \delta \int_a^b (\mathcal{L} + \lambda g) dt = 0$$

- Semi-Holonomic Constraint

$$g(\dot{\mathbf{q}}_i) = \sum_{i \in S} \mathbf{p}_i - M_I \mathbf{u}_I = 0$$



- Constraint Equation of O'Connell and Thompson (1995)

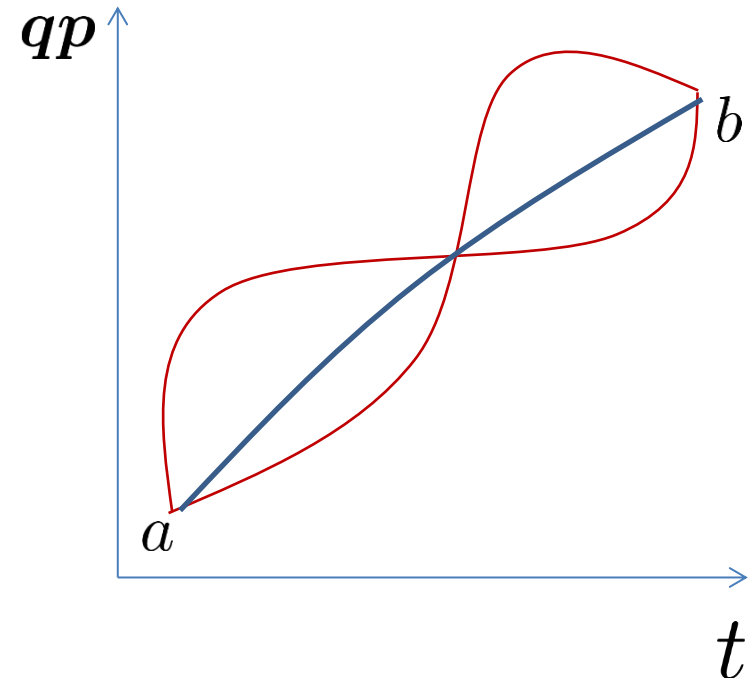
$$m_i \dot{\mathbf{q}}_i = \mathbf{p}_i - \frac{1}{m N_I} \left[\sum_{i \in S} \mathbf{p}_i - M_I \mathbf{u}_I \right]$$

$$\dot{\mathbf{p}}_i = \mathbf{F}_i$$

Constraint Algorithms

- Principle of Least Action (subject to constraint)

$$\delta A_c = \delta \int_a^b (\mathcal{L} + \lambda g) dt = 0$$



- Non-Holonomic Constraint

$$g(\mathbf{q}_i, \mathbf{p}_i) = \sum_{i=1}^N \mathbf{p}_i \vartheta_i - \int_V \rho \mathbf{u} dV = 0$$

- Constraint Equation

$$\left. \begin{aligned} m_i \dot{\mathbf{q}}_i &= \mathbf{p}_i - \frac{m_i \vartheta_i}{M_I} \left[\sum_{n=1}^N \mathbf{p}_n \vartheta_n - \int_V \rho \mathbf{u} dV \right] \\ \dot{\mathbf{p}}_i &= \mathbf{F}_i \end{aligned} \right\} m_i \ddot{\mathbf{q}}_i = \mathbf{F}_i + \mathbf{F}_i^C$$

Where $M_I \equiv \sum_{n=1}^N m_n \vartheta_n$

Constraint Algorithms

- Combining the CV equations of O'Connell and Thompson

$$\begin{aligned}
 m_i \ddot{\mathbf{q}}_i = & \mathbf{F}_i + \frac{m_i \vartheta_i}{M_I} \left[\frac{d}{dt} \int_V \rho \mathbf{u} dV - \underbrace{\sum_{n,m}^N \mathbf{f}_{nm} \vartheta_{nm}}_{\mathbf{F}_{\text{surface}}} + \underbrace{\sum_{n=1}^N m_i \dot{\mathbf{q}}_n \dot{\mathbf{q}}_n \cdot d\mathbf{S}_n}_{\text{Momentum Flux}} \right] \\
 & + \frac{m_i}{M_I} \left(\underbrace{\dot{\mathbf{q}}_i \cdot d\mathbf{S}_i - \frac{\vartheta_i}{M_I} \sum_{n=1}^N m_n \dot{\mathbf{q}}_n \cdot d\mathbf{S}_n}_{\text{Mass Flux}} \right) \times \underbrace{\left[\sum_{n=1}^N m_n \dot{\mathbf{q}}_n \vartheta_n - \int_V \rho \mathbf{u} dV \right]}_{\text{Momentum Discrepancy}}
 \end{aligned}$$

- Surface forces only are included in the constraint
 - The CV is effectively isolated (no forces) from the rest of the system
- Extra terms due to molecular flux
 - Molecules crossing the surface result in extra forces

Constraint Algorithms

- Combining the CV equations of O'Connell and Thompson

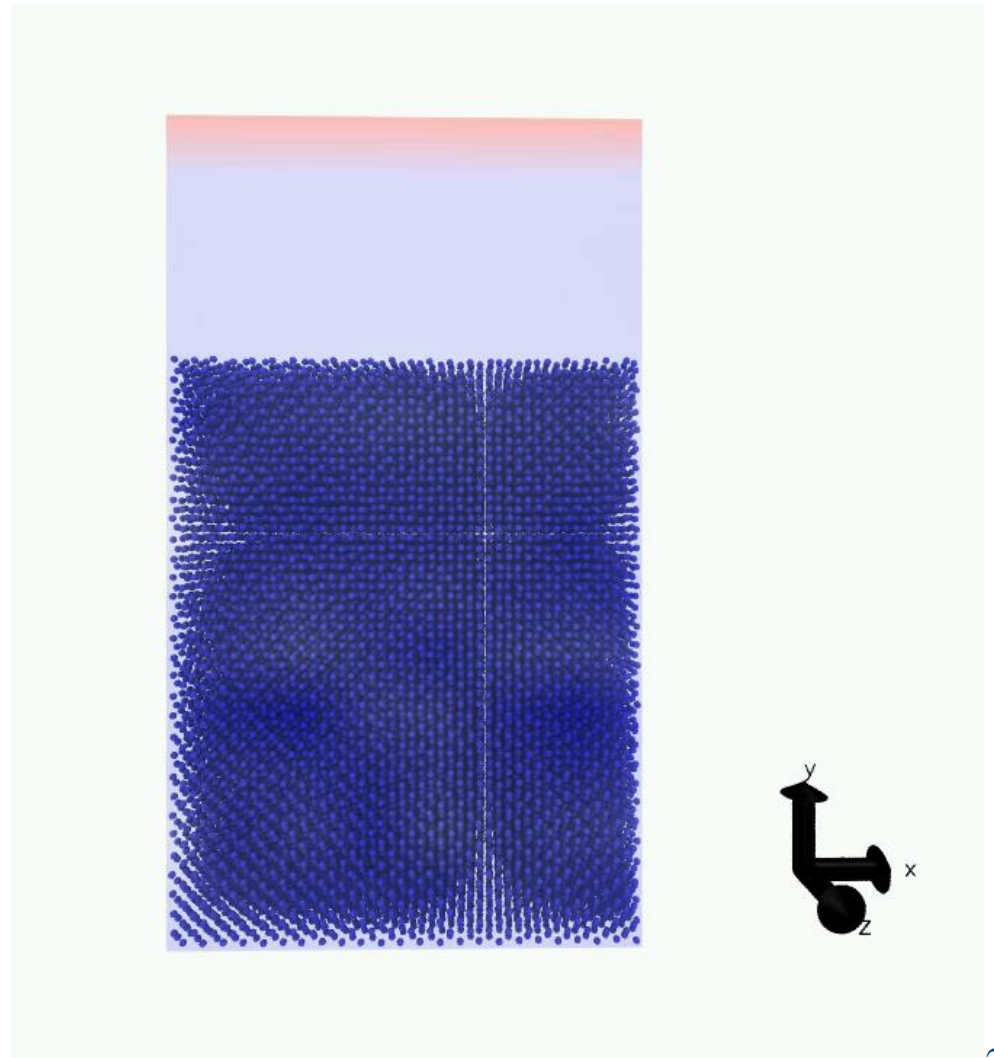
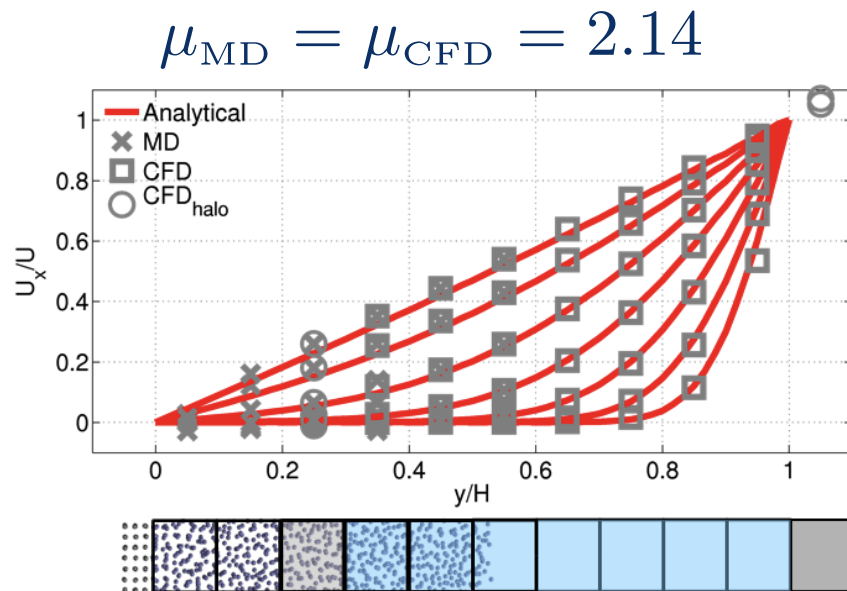
$$\begin{aligned}
 m_i \ddot{\mathbf{q}}_i = & \mathbf{F}_i + \frac{m_i \vartheta_i}{M_I} \left[\frac{d}{dt} \int_V \rho \mathbf{u} dV - \overbrace{\sum_{n,m}^N \mathbf{f}_{nm} \vartheta_{nm}}^{\mathbf{F}_{\text{surface}}} + \overbrace{\sum_{n=1}^N m_i \dot{\mathbf{q}}_n \dot{\mathbf{q}}_n \cdot d\mathbf{S}_n}^{\text{Momentum Flux}} \right] \\
 & + \frac{m_i}{M_I} \left(\underbrace{\dot{\mathbf{q}}_i \cdot d\mathbf{S}_i - \frac{\vartheta_i}{M_I} \sum_{n=1}^N m_n \dot{\mathbf{q}}_n \cdot d\mathbf{S}_n}_{\text{Mass Flux}} \right) \times \underbrace{\left[\sum_{n=1}^N m_n \dot{\mathbf{q}}_n \vartheta_n - \int_V \rho \mathbf{u} dV \right]}_{\text{Momentum Discrepancy}}
 \end{aligned}$$

- Negligible surface flux and average mass/velocity inside volume yields Nie, Chen, E and Robbins (2004) coupling

$$m \ddot{\mathbf{q}}_i = \mathbf{F}_i + m \vartheta_i \left[\frac{D\mathbf{u}_I}{Dt} - \frac{1}{m N_I} \sum_{n=1}^N \mathbf{F}_n \vartheta_n \right]$$

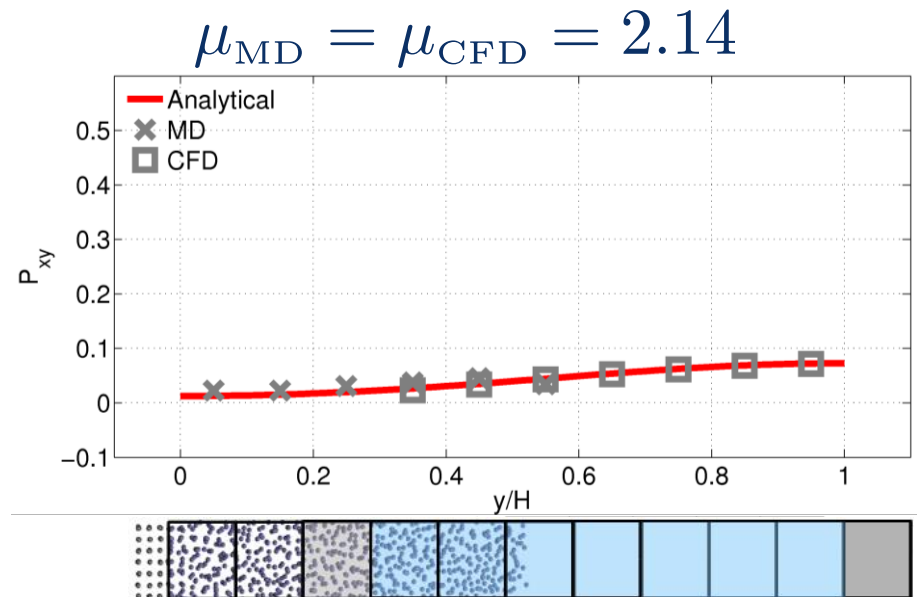
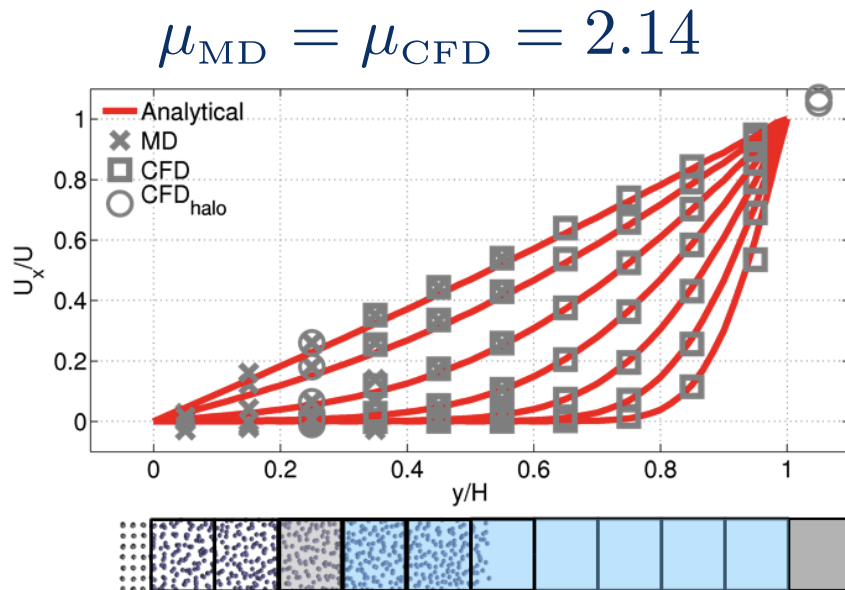
State Coupling

- Continuum-MD coupling using state variables shows good agreement in the velocity profiles



State Coupling

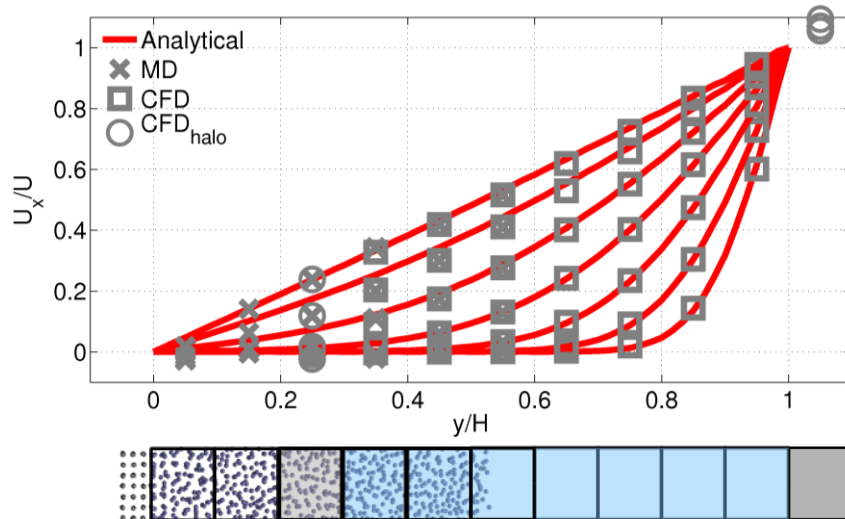
- Continuum-MD coupling using state variables shows good agreement in the velocity profiles



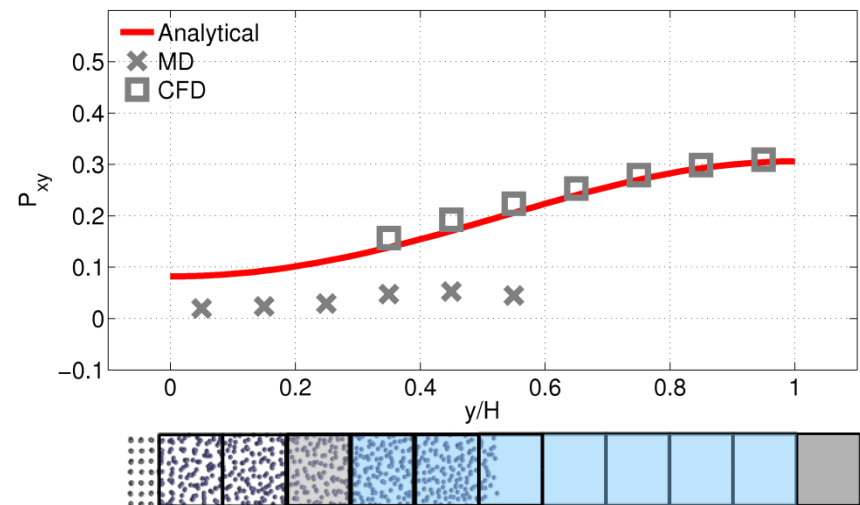
State Coupling

- Continuum-MD coupling using state variables shows good agreement in the velocity profiles
 - However, a mismatch in the stress can occur at the interface
 - Unless both regions have a constant viscosity, fluxes are not coupled despite agreement of velocity profile

$$\mu_{\text{MD}} = 2.14 \quad \mu_{\text{CFD}} = 10$$



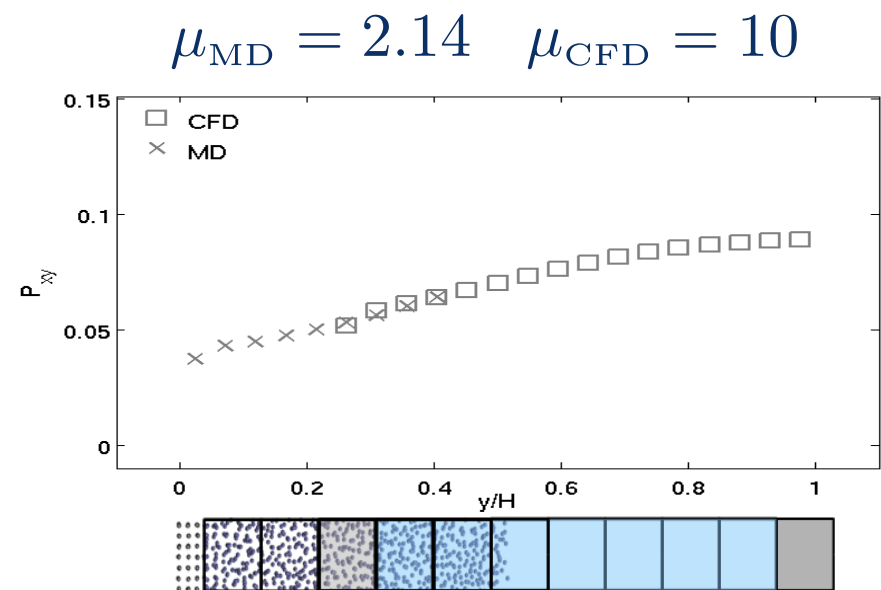
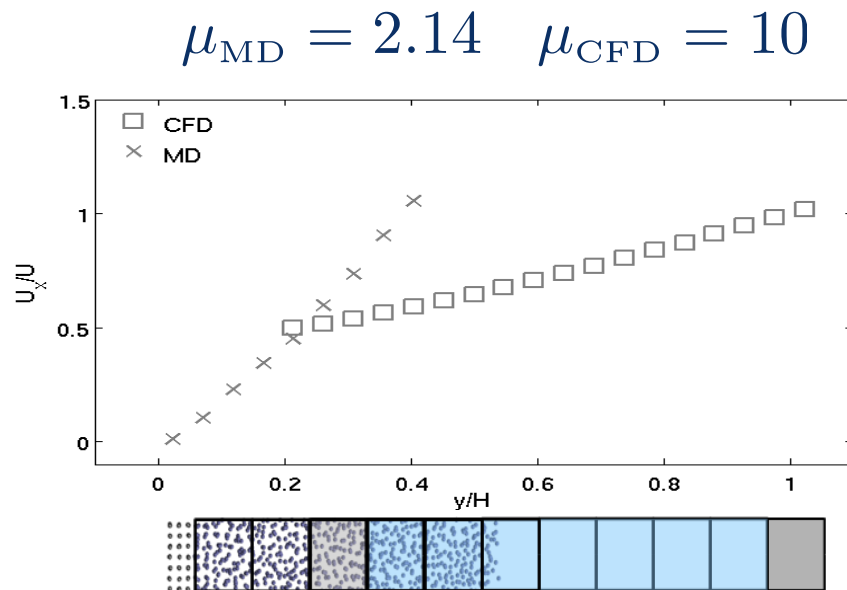
$$\mu_{\text{MD}} = 2.14 \quad \mu_{\text{CFD}} = 10$$



- Flux coupling is required

Flux Coupling

- Continuum-MD coupling using flux variables shows good agreement in the stress profiles*
 - A mismatch in the velocity occurs at the interface
 - Analogous to the interface between two immiscible fluids



- Flux Coupling requires significant statistical averaging
 - Coupling developed for massively parallel applications to exploit high performance computers
- *E. G. Flekkøy, G. Wagner, and J. Feder, Europhys. Lett. 52, 271 (2000).

Flux Coupling and the Pressure Tensor

Flux Coupling

- Flux Coupling requires the equations of motion in terms of stresses and fluxes

Surface Forces

$$\frac{\partial}{\partial t} \int_V \rho \mathbf{u} dV = - \oint_S \rho \mathbf{u} \mathbf{u} \cdot d\mathbf{S} + \mathbf{F}_{\text{surface}}$$

Divergence of stress

$$\frac{\partial}{\partial t} \int_V \rho \mathbf{u} dV = - \int_V \frac{\partial}{\partial \mathbf{r}} \cdot \rho \mathbf{u} \mathbf{u} - \int_V \frac{\partial}{\partial \mathbf{r}} \cdot [P\mathbf{I} - \boldsymbol{\sigma}] dV$$

Surface Stresses

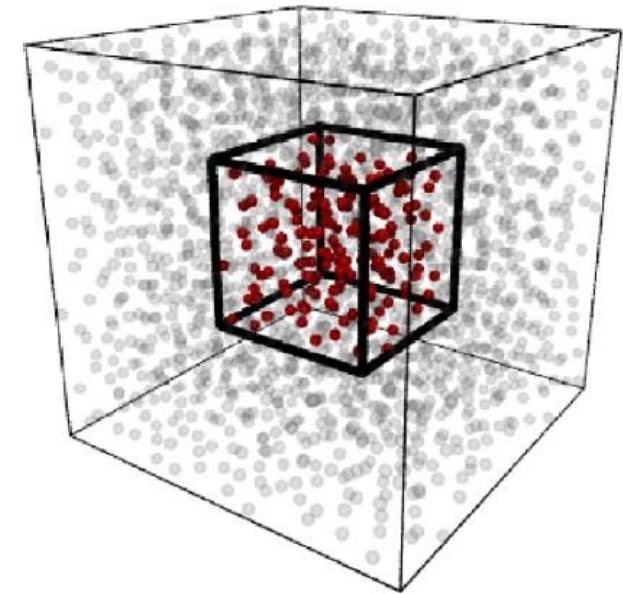
$$\frac{\partial}{\partial t} \int_V \rho \mathbf{u} dV = - \oint_S \rho \mathbf{u} \mathbf{u} \cdot d\mathbf{S} - \oint_S [P\mathbf{I} - \boldsymbol{\sigma}] \cdot d\mathbf{S}$$



Control Volume Function

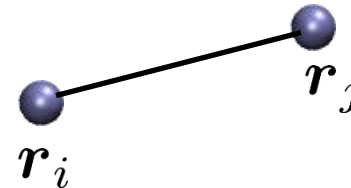
- The Control volume function is the integral of the Dirac delta function in 3 dimensions

$$\begin{aligned}\vartheta_i &\equiv \int_V \delta(\mathbf{r} - \mathbf{r}_i) dV \\ &= [H(x^+ - x_i) - H(x^- - x_i)] \\ &\quad \times [H(y^+ - y_i) - H(y^- - y_i)] \\ &\quad \times [H(z^+ - z_i) - H(z^- - z_i)]\end{aligned}$$



- Replace molecular position with equation for a line

$$\mathbf{r}_i \rightarrow \mathbf{r}_i - s\mathbf{r}_{ij}$$



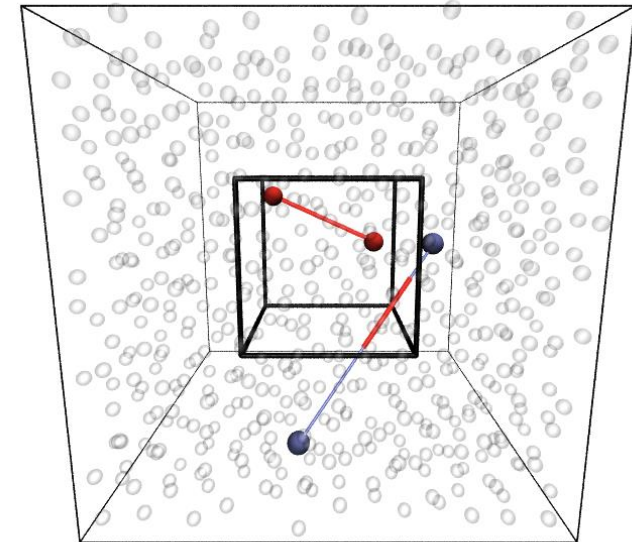
Control Volume Function

- The Control volume function is the integral of the Dirac delta function in 3 dimensions

$$\vartheta_s \equiv \int_V \delta(\mathbf{r} - \mathbf{r}_i + s\mathbf{r}_{ij}) dV =$$
$$\left[H(x^+ - x_i + sx_{ij}) - H(x^- - x_i + sx_{ij}) \right]$$
$$\times \left[H(y^+ - y_i + sy_{ij}) - H(y^- - y_i + sy_{ij}) \right]$$
$$\times \left[H(z^+ - z_i + sz_{ij}) - H(z^- - z_i + sz_{ij}) \right]$$

- Length of interaction inside the CV

$$l_{ij} = \int_0^1 \vartheta_s ds$$



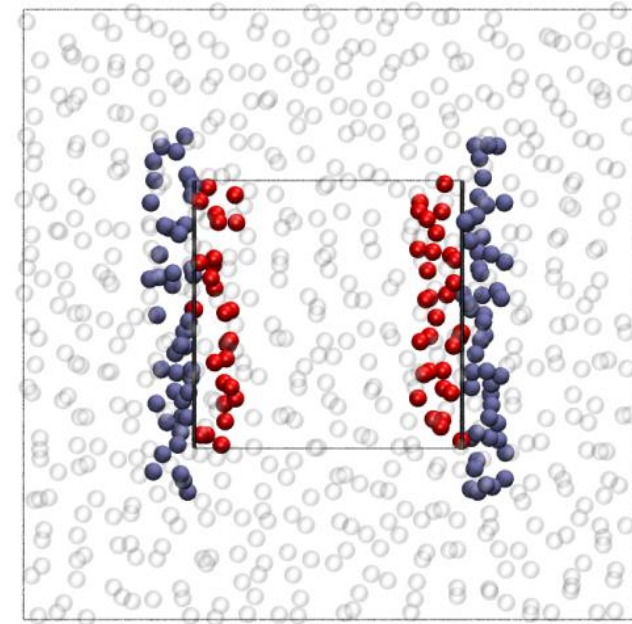
Derivatives Yield the Surface Forces

- Taking the Derivative of the CV function

$$\begin{aligned} \frac{\partial \vartheta_s}{\partial \mathbf{r}} &\equiv \left[\delta(x^+ - x_i + s x_{ij}) - \delta(x^- - x_i + s x_{ij}) \right] \\ &\times \left[H(y^+ - y_i + s y_{ij}) - H(y^- - y_i + s y_{ij}) \right] \\ &\times \left[H(z^+ - z_i + s z_{ij}) - H(z^- - z_i + s z_{ij}) \right] \end{aligned}$$

- Surface fluxes over the top and bottom surface

$$\tilde{\mathbf{n}} \cdot d\mathbf{S}_{ij} = \int_0^1 \frac{\partial \vartheta_s}{\partial \mathbf{r}} ds \quad \tilde{\mathbf{n}} = [1 \ 1 \ 1]$$



Surface Pressures

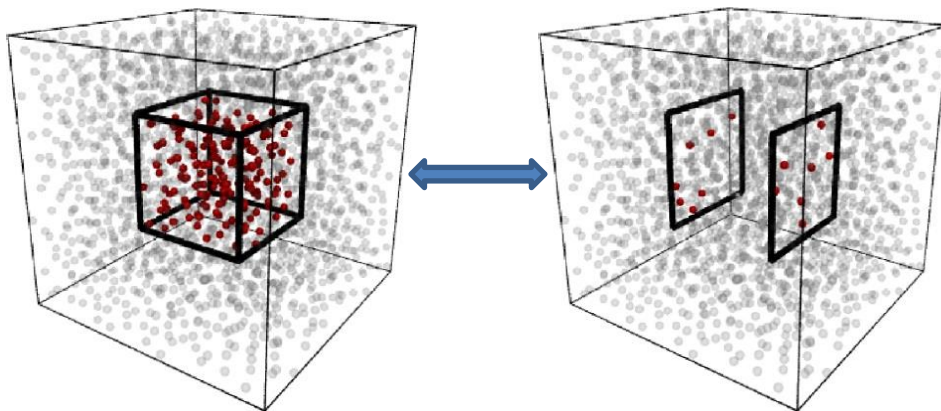
- Volume Average Form of Lutsko (1988) & Cormier et al (2001)

$$\int_V \frac{\partial}{\partial \mathbf{r}} \cdot [P\mathbf{I} - \boldsymbol{\sigma}] dV = \frac{\partial}{\partial \mathbf{r}} \cdot \sum_{i=1}^N \left[(\mathbf{v}_i - \mathbf{u})(\mathbf{v}_i - \mathbf{u}) \vartheta_i + \sum_{j \neq i}^N \mathbf{f}_{ij} \mathbf{r}_{ij} \int_0^1 \vartheta_s ds \right]$$

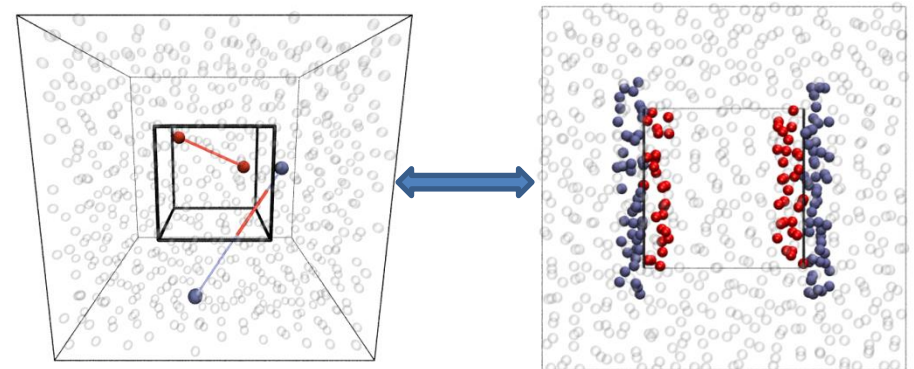
- The derivative of ϑ_s gives forces over the surface - a localisation of the method of planes, (Todd et al 1995, Han Lee 2004)

$$\oint_S [P\mathbf{I} - \boldsymbol{\sigma}] \cdot d\mathbf{S} = \sum_{i=1}^N \left[(\mathbf{v}_i - \mathbf{u})(\mathbf{v}_i - \mathbf{u}) \cdot d\mathbf{S}_i + \sum_{j \neq i}^N \mathbf{f}_{ij} \tilde{\mathbf{n}} \cdot d\mathbf{S}_{ij} \right]$$

- Kinetic



- Configurational



Surface Pressures

- **Volume Average Form of Lutsko (1988) & Cormier et al (2001)**

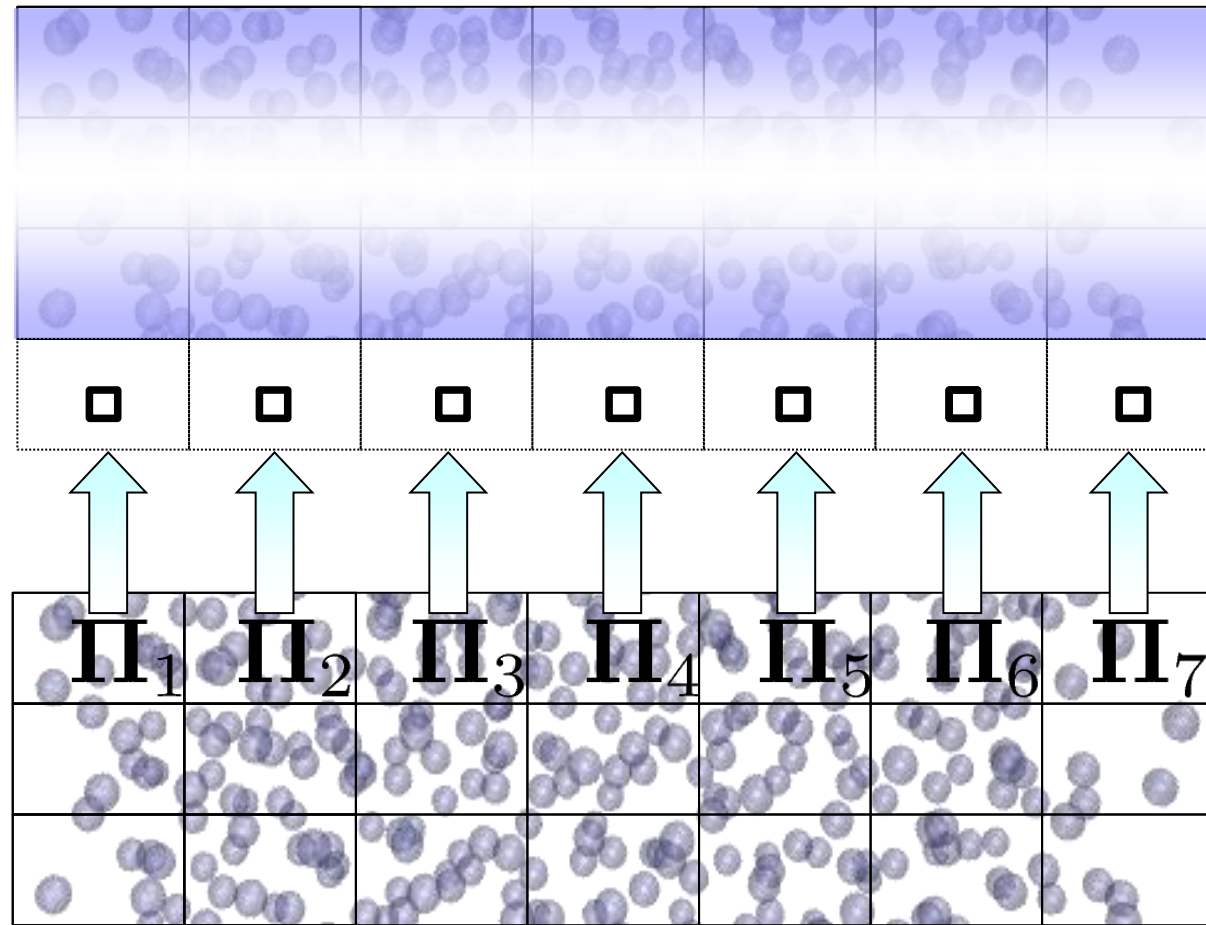
$$\int_V \frac{\partial}{\partial \mathbf{r}} \cdot [P\mathbf{I} - \boldsymbol{\sigma}] dV = \frac{\partial}{\partial \mathbf{r}} \cdot \sum_{i=1}^N \left[(\mathbf{v}_i - \mathbf{u})(\mathbf{v}_i - \mathbf{u}) \vartheta_i + \sum_{j \neq i}^N \mathbf{f}_{ij} \mathbf{r}_{ij} \int_0^1 \vartheta_s ds \right]$$

- **The derivative of ϑ_s gives forces over the surface - a localisation of the method of planes, (Todd et al 1995, Han Lee 2004)**

$$\oint_S [P\mathbf{I} - \boldsymbol{\sigma}] \cdot d\mathbf{S} = \sum_{i=1}^N \left[(\mathbf{v}_i - \mathbf{u})(\mathbf{v}_i - \mathbf{u}) \cdot d\mathbf{S}_i + \sum_{j \neq i}^N \mathbf{f}_{ij} \tilde{\mathbf{n}} \cdot d\mathbf{S}_{ij} \right]$$

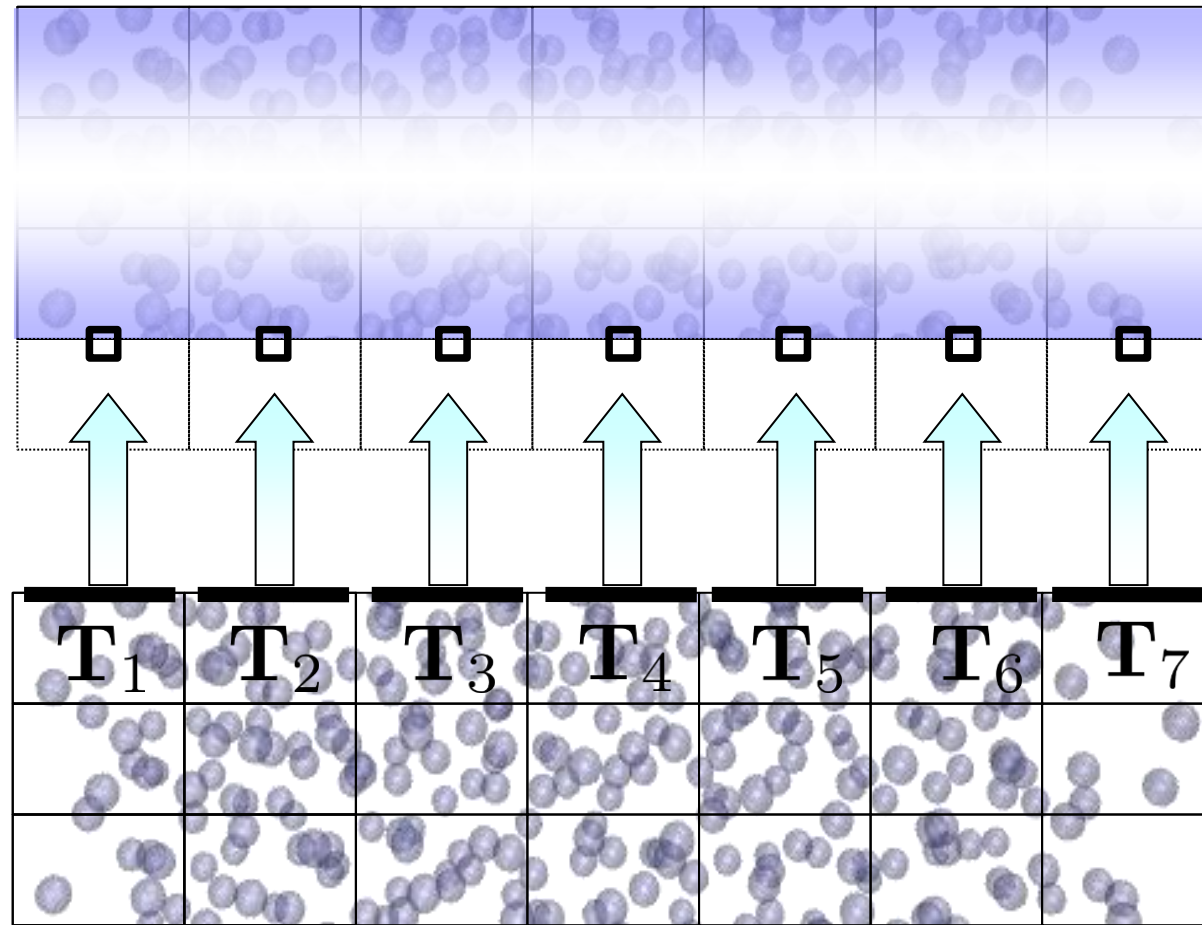
- **Stress tensor is non-unique but link shown between**
 - Volume Average (Lutsko, 1988) and Method of Planes (Todd et al 1995)
- **Exact relationship between surface flux/stress form and momentum change inside the control volume**
 - Conservation can be used to obtain unknown values
 - Most appropriate form of the non-unique stress tensor for coupling

Coupling the Localised Stresses



$$\mathbf{\Pi} = \sum_{i=1}^N \left[(\mathbf{v}_i - \mathbf{u})(\mathbf{v}_i - \mathbf{u}) \vartheta_i + \sum_{j \neq i}^N \mathbf{f}_{ij} \mathbf{r}_{ij} \int_0^1 \vartheta_s ds \right]$$

Coupling the Surface Stresses



$$\mathbf{T} = \sum_{i=1}^N \left[(\mathbf{v}_i - \mathbf{u}) (\mathbf{v}_i - \mathbf{u}) \cdot d\mathbf{S}_{yi}^+ + \sum_{j \neq i}^N \mathbf{f}_{ij} \tilde{\mathbf{n}} \cdot d\mathbf{S}_{ij}^+ \right]$$

State or Flux Constraint?

- Recall the coupled CV equations

$$\begin{aligned}
 m_i \ddot{\mathbf{q}}_i &= \mathbf{F}_i + \frac{m_i \vartheta_i}{M_I} \left[\frac{d}{dt} \int_V \rho \mathbf{u} dV - \underbrace{\sum_{n,m}^N \mathbf{f}_{nm} \vartheta_{nm}}_{\mathbf{F}_{\text{surface}}} + \underbrace{\sum_{n=1}^N m_i \dot{\mathbf{q}}_n \dot{\mathbf{q}}_n \cdot d\mathbf{S}_n}_{\text{Momentum Flux}} \right] \\
 &\quad + \frac{m_i}{M_I} \left(\underbrace{\dot{\mathbf{q}}_i \cdot d\mathbf{S}_i - \frac{\vartheta_i}{M_I} \sum_{n=1}^N m_n \dot{\mathbf{q}}_n \cdot d\mathbf{S}_n}_{\text{Mass Flux}} \right) \times \underbrace{\left[\sum_{n=1}^N m_n \dot{\mathbf{q}}_n \vartheta_n - \int_V \rho \mathbf{u} dV \right]}_{\text{Momentum Discrepancy}} - \oint_S [\rho \mathbf{u} \mathbf{u} + P \mathbf{I} - \boldsymbol{\sigma}] \cdot d\mathbf{S}
 \end{aligned}$$

- Includes both force and fluctuation terms*
- Energy or Entropy CV equation could also be analysed/applied in a similar manner

• *

*Flekkoy, Delgado-Buscalioni and Coveney (2005)

Control Volume Coupling

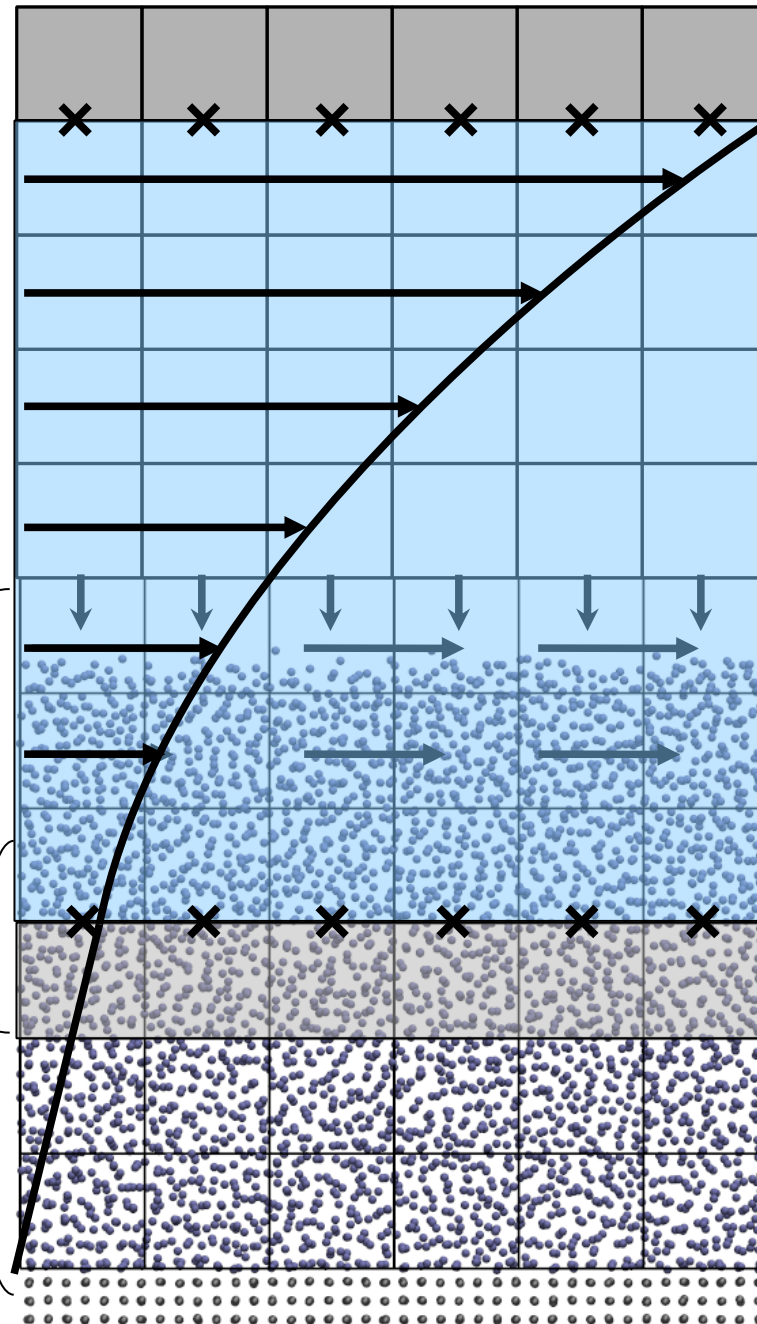
- Molecular Equations**

$$\frac{d}{dt} \sum_{i=1}^N m_i \mathbf{v}_i \vartheta_i = - \oint_S \rho \mathbf{u} \mathbf{u} \cdot d\mathbf{S}$$

$$- \sum_{i=1}^N (\mathbf{v}_i - \mathbf{u}) (\mathbf{v}_i - \mathbf{u}) \cdot d\mathbf{S}_i$$

$$- \sum_{i=1}^N \sum_{j \neq i}^N \mathbf{S}_{ij} \cdot d\mathbf{S}_{ij}$$

$$m \ddot{\mathbf{r}}_i = \sum_{j \neq i}^N \mathbf{f}_{ij}$$



$$\rho u = \rho U_0$$

$$\frac{\partial}{\partial t} \int_V \rho u dV = - \oint_S \rho u \mathbf{u} \cdot d\mathbf{S} - \oint_S \mathbf{\Pi} \cdot d\mathbf{S}$$

- Continuum Equations**

Summary

- **Introduced a novel mathematical function to define a control volume in a discrete system**
 - Derived in a manner consistent with a continuum control volume
 - Mathematically well defined and applicable to any discrete system
- **Derivation of discrete CV conservation equation**
 - Allows control volume analysis to be extended to nano-scale systems
 - The resulting equations are exactly conservative in a discrete system
- **Application to Coupling**
 - Allows both systems to be written in a consistent framework
 - Averages from molecular region can be written in terms of continuum variables, e.g. surface fluxes
 - Facilitates derivation of constraint algorithms using minimisation principles

References

• References

J. H. Irving and J. G. Kirkwood, J. Chemical Phys. 18(6), 817 (1950).

R.J. Hardy J. Chem. Phys. 76, 1 (1998)

P. Schofield and J. R. Henderson, Proc. R. Soc. London A 379, 231 (1982).

J. F. Lutsko, J. Appl. Phys 64(3), 1152 (1988)

S. T. O'Connell and P. A. Thompson, Phys. Rev. E 52, R5792 (1995).

B. D. Todd, D. J. Evans, and P. J. Daivis, Physical Review E 52(2), 1627 (1995).

N. G. Hadjiconstantinou, Ph.D. thesis, MIT, 1998.

E. G. Flekkøy, G. Wagner, and J. Feder, Europhys. Lett. 52, 271 (2000).

J. Cormier, J. Rickman, and T. Delph, J. Appl. Phys 89-1, 99 (2001).

M. Han and J. Lee, Phys. Rev. E 70, 061205 (2004).

X. B. Nie, S. Y. Chen, W. N. E, and M. O.

Robbins, J. Fluid Mech. 500, 55 (2004)
R. Delgado-Buscalioni and P. Coveney, Phil. Trans. R. Soc. Lond. 362, 1639 (2004).

E. G. Flekkoy, R. Delgado-Buscalioni and P.V. Coveney ,Flux Boundary Conditions in Particle Simulation, Phys. Rev. E, **72**, 026703 (2005)

Wm. G. Hoover, C. G. Hoover Phys. Rev. E 80 011128 (2009)

A. I. Murdoch, J. Elast. 100, 33 (2010).

D. M. Heyes, E. R. Smith, D. Dini, T. A. Zaki J. Chemical Phys. 135, 024512 (2011)

E.R. Smith, D.M. Heyes, D. Dini, T.A. Zaki, Phys. Rev. E 85. 056705 (2012)

Thank you for listening

Any Questions?

Future Aims

- **We have the mathematical framework to express both systems consistently**
 - Matching control volumes in both regions
 - Surface fluxes and stress in the CFD and MD are equivalent
- **Apply the extended CV constraint equations to derived exactly conservative coupling**
 - Total coupled system is exactly conservative
 - Energy and entropy changes can be evaluated exactly
- **Large scale coupling simulation**
 - Flexible computational coupling framework already developed for parallel MD and CFD codes
 - Simulate very large and highly non-linear problems using coupled molecular and continuum systems

Moving reference frame

- Why the continuum form of Reynolds' transport theorem has a partial derivative but the discrete is a full derivative

- Eulerian mass conservation

$$\vartheta_i = \vartheta_i(\mathbf{r}_i(t), \mathbf{r})$$

$$\frac{d}{dt} \sum_{i=1}^N m_i \vartheta_i = - \sum_{i=1}^N m_i \mathbf{v}_i \cdot d\mathbf{S}_i$$

$$\frac{\partial}{\partial t} \int_V \rho dV = - \oint_S \rho \mathbf{u} \cdot d\mathbf{S}$$

- Lagrangian mass conservation

$$\vartheta_i = \vartheta_i(\mathbf{r}_i(t), \mathbf{r}(t))$$

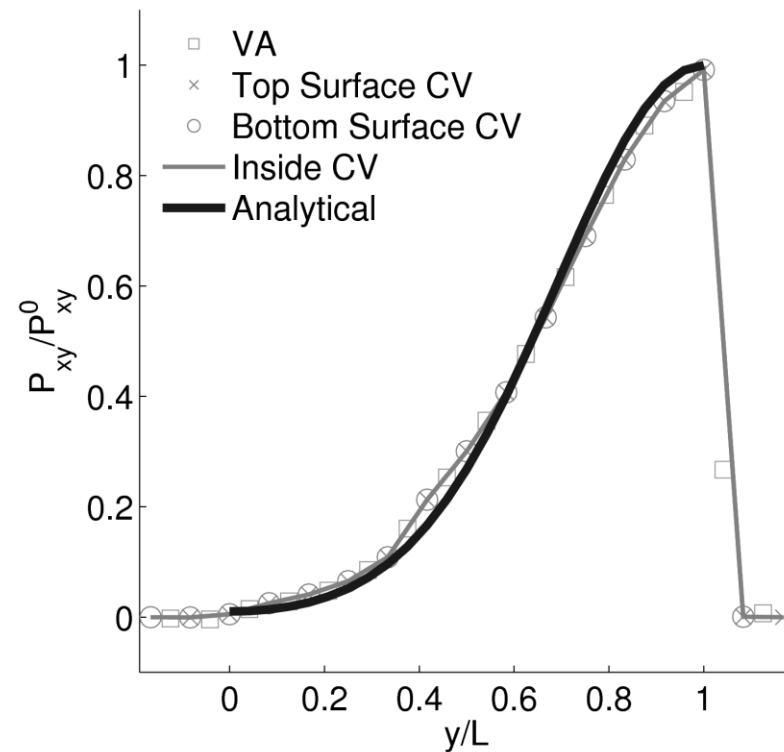
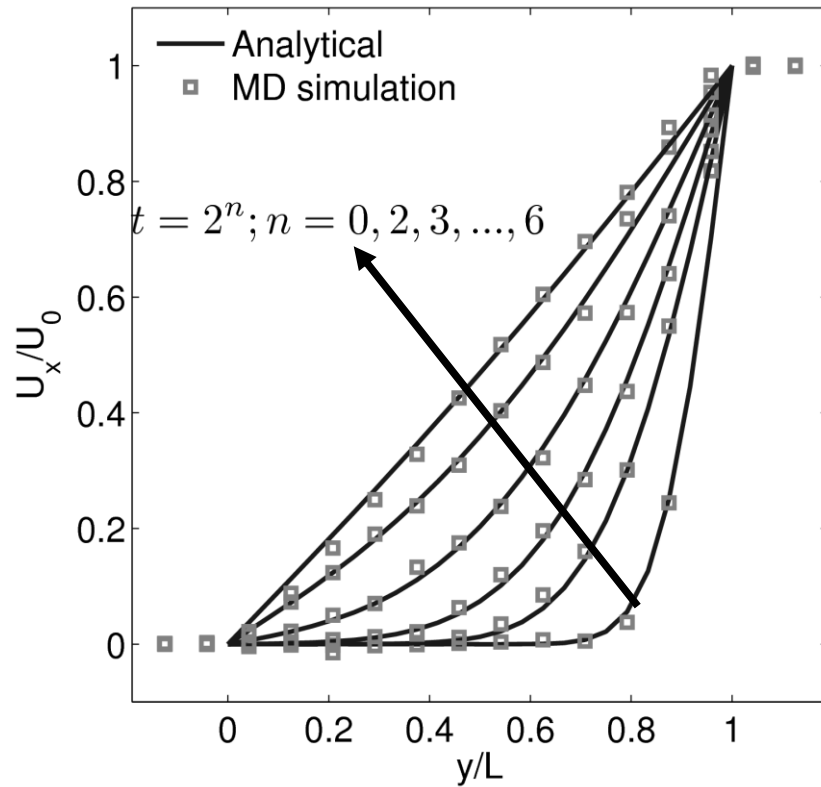
$$\frac{d}{dt} \sum_{i=1}^N m_i \vartheta_i = - \sum_{i=1}^N m_i (\mathbf{v}_i + \bar{\mathbf{u}}) \cdot d\mathbf{S}_i$$

$$\frac{d}{dt} \int_V \rho dV = \oint_S \rho (\mathbf{u} - \bar{\mathbf{u}}) \cdot d\mathbf{S}$$

$$\bar{\mathbf{u}} \cdot d\mathbf{S}_i = \frac{d\mathbf{r}}{dt} \cdot \frac{d\vartheta_i}{d\mathbf{r}}$$

$$\oint_S \rho \mathbf{u} \cdot d\mathbf{S} - \oint_S \rho \bar{\mathbf{u}} \cdot d\mathbf{S} = 0$$

Continuum Analytical Couette Flow



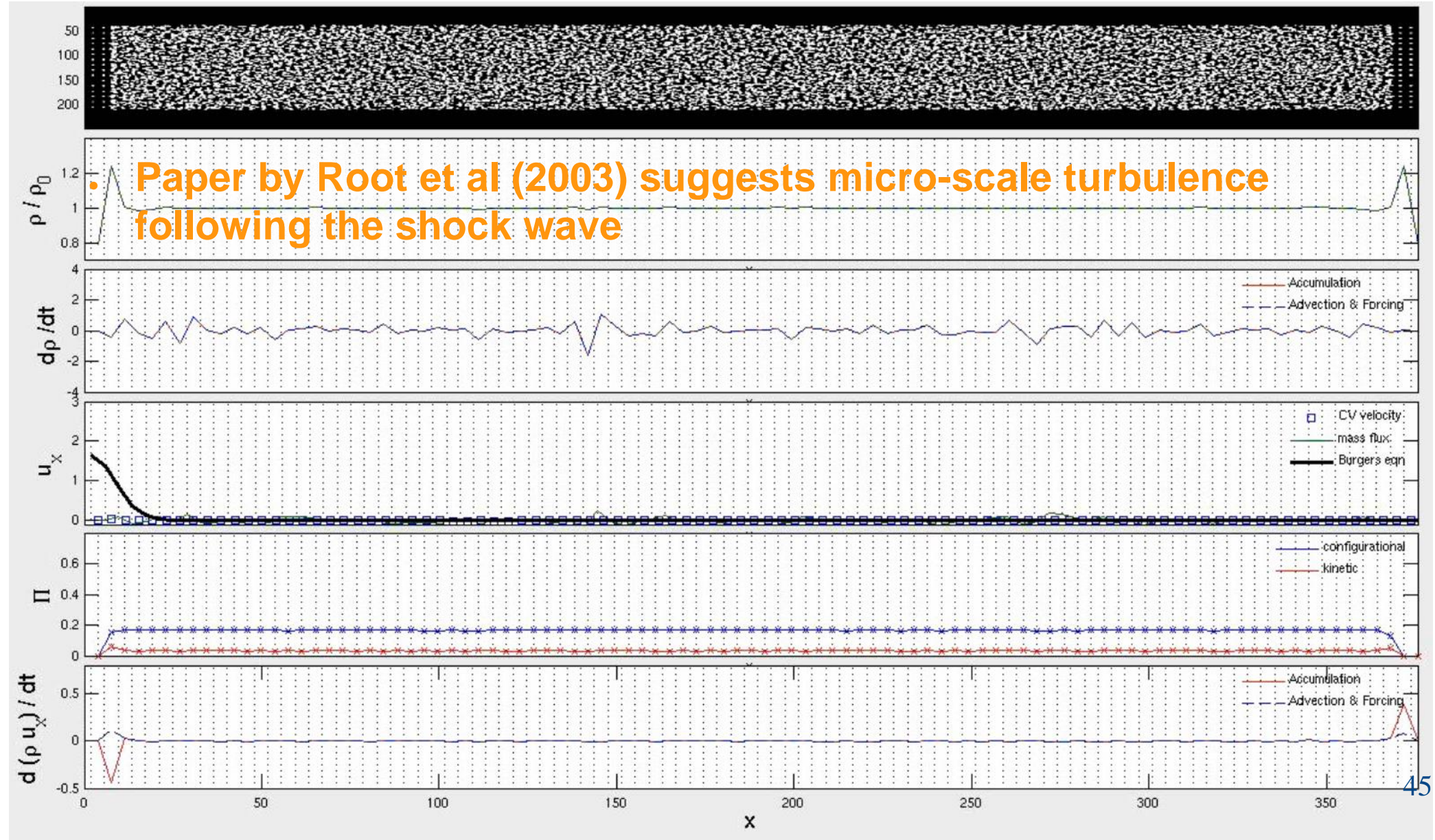
$$u_x(y, t) = \begin{cases} U_0 & y = L \\ \sum_{n=1}^{\infty} u_n(t) \sin\left(\frac{n\pi y}{L}\right) & 0 < y < L \\ 0 & y = 0 \end{cases}$$

$$\Pi_{xy}(y, t) = \frac{\mu U_0}{L} \left[1 + 2 \sum_{n=1}^{\infty} (-1)^n e^{-\frac{\lambda_n \mu t}{\rho}} \cos\left(\frac{n\pi y}{L}\right) \right]$$

Where, $\lambda_n = \left(\frac{n\pi}{L}\right)^2$ and $u_n(t) = \frac{2U_0(-1)^n}{n\pi} \left(e^{-\frac{\lambda_n \mu t}{\rho}} - 1\right)$

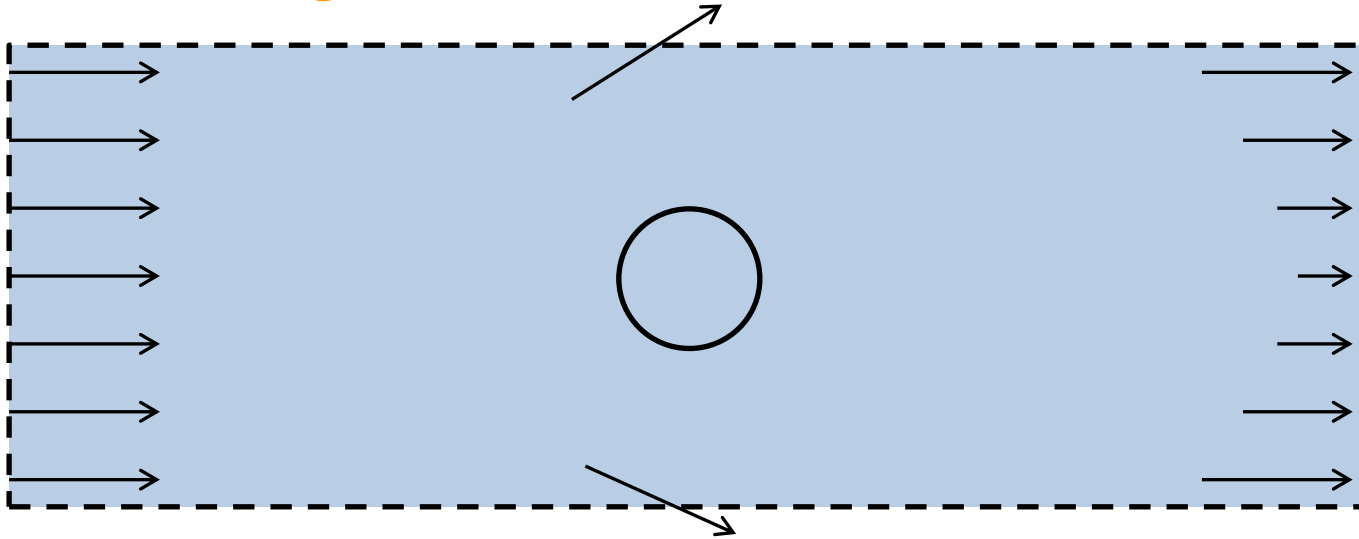
Shockwaves

- Current work on application of control volume theory



Flow past a cylinder

- Use of the momentum conservation of the control volume to determine the drag coefficient



- Drag over a Carbon Nano-tube can be determined

